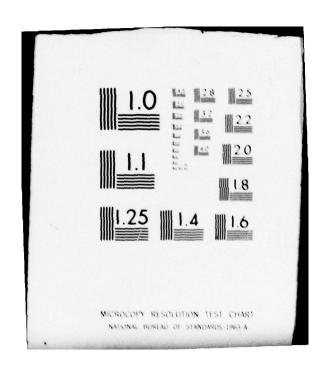
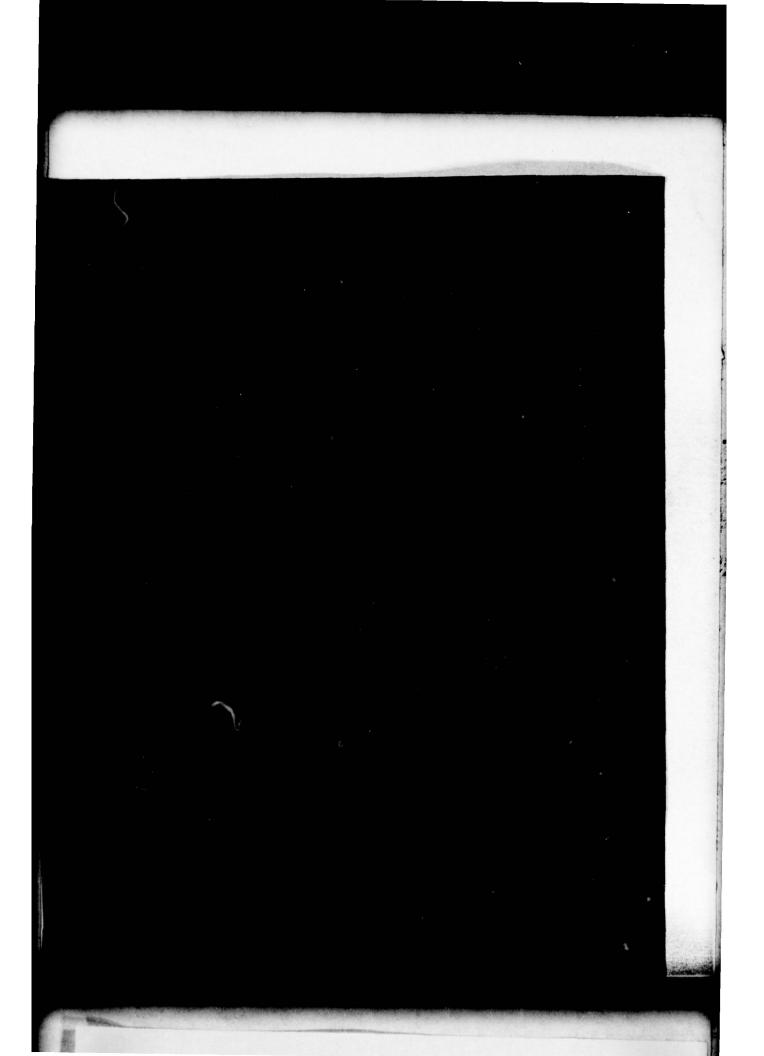
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FOREWORD

The icebreaking operations conducted by the U.S. Coast
Guard on the Illinois and Mississippi Rivers during the
1977-1978 winter season were designed to evaluate the applicability of air cushion vehicles as icebreakers in a river
environment. Two air cushion vehicles were employed in this
project, a nonself-propelled barge craft (RIVER GUARDIAN) and
a self-propelled craft (LACV-30). It was not within the scope
of the study to perform any comparative evaluation of icebreaking effectiveness between the two craft, and field
operations were not designed for this purpose. The operational
procedures for each craft have been discussed separately and
results obtained have been explained under separate headings
for clarity. Comparative data can only be developed through
further research and evaluation involving the two craft.

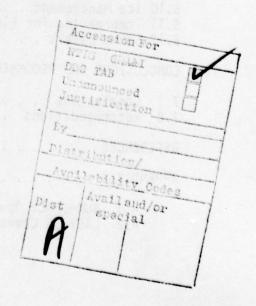


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ACK NOWLEDGEMENTS

The technical direction and encouragement provided in this project by

CDR Robert H. Cassis, Jr., of Coast Guard Headquarters Office of Research and

Development and Messrs. Clark W. Pritchett and Robert E. Williams of the Coast

Buard Research and Development Center are greatly appreciated.

In addition, the efforts of the following Commands and Organizations and their contributions to the test and evaluation are gratefully acknowledged:

Second Coast Guard District and participating units and commands

- U.S. Army, MERADCOM, Fort Belvoir, Virginia
- U.S. Army, Transportation School, Fort Eustis, Virginia
- U.S. Army, CRREL, Hanover, New Hampshire
- U.S. Army, Corps of Engineers, Chicago District and Peoria and Joliet project offices

ADMINISTRATIVE INFORMATION

In response to the Chief, Office of Operations, the Office of Research and Development (G-D) was assigned the responsibility of conducting a Test and Evaluation Program to determine the feasibility of using air cushion vehicles (ACV's) as icebreakers in a river ice environment (Reference 1). The Chief of Operations forwarded a task statement to the Chief, Office of Research and Development (Reference 2), who assigned the responsibility for developing the Test Plan, conducting the operational and engineering tests and preparing the final documentation to the Research and Development Center (Reference 3).

To carry out the ACV icebreaking operational/engineering test and evaluation program, two ACV craft, a nonself-propelled ACV barge and the self-propelled LACV-30, were obtained. The barge-type ACV was obtained under a lease contract from Mackley Ace (USA), Inc., for the period of 1 December 1977 to 28 February 1978 (Reference 4). The LACV-30 was obtained through the cooperation of the U.S. Army through a memorandum of understanding document for the approximate period of 15 November 1977 to 15 March 1978 (Reference 5).

Engineering support services were obtained by the Research and Development Center through a contract with the firm of Chi Associates, Inc., Arlington, Virginia, who employed as part of its team a subcontractor, Giannotti and Buck, Inc., Riverdale, Maryland (Reference 6).

The ACV icebreaking operational/engineering test and evaluation program was conducted on the Illinois and Mississippi Rivers with operations centered around Peoria, Illinois, in accordance with OPORDER CG 2-78. The operational base was the Coast Guard ANFAC (Aids to Navigation Facility) in

East Peoria within the Second Coast Guard District.

The following personnel were on site during the test and evaluation program:

- Clark W. Pritchett, Project Manager, U.S. Coast Guard Research and Development Center
- Lt. John Anthony, U.S. Coast Guard, LACV-30 Test Director, G-DOE-4
- o Orin Stark, SUMAC/ACV Test Director, U.S. Coast Guard Research and Development Center
- o CWO F.X. McCarthy, SUMAC/ACV Test Director, U.S. Coast Guard Research and Development Center
- Jon Buck, Technical Engineering Support/Test Engineer, Giannotti and Buck, Inc.
- Bernard Dennis, Technical Engineering Support/Test Engineer, Chi Associates, Inc.
- o CWO H. Sites, U.S. Coast Guard Research and Development Center
- o CWO George Matheson, U.S. Coast Guard, CGC SUMAC Commanding Officer
- o Crew of U.S. Coast Guard Cutter SUMAC
- o Major Lyle Haskins, U.S. Army, Commander of LACV-30
- o LACV-30 Crew and Support Group
- o Field Personnel from Mackley Ace, Inc. Ray Golding, ACV Supervisor, 12-1-77 to 1-1-78 Alan Brice, ACV Supervisor, 1-1-78 to 3-8-78 John Latimer, Technical Engineer Keith Jenman, Technical Engineer
- o Field visits were made by representatives of CRREL.

ODE TO THE HOVERCRAFT*

Hark, what is that mighty roar? Rising, bellowing, more and more: "'Tis the Hovercraft," they shout, "Here to break the ice and move it out."

O wonderful Hovercraft, save our trailers, Show us all that we're not failures. Break the ice and stir the waters. Save our sons and save our daughters.

With multitudes upon the bank assembled. The sky it darkened, the ground it trembled. The ice broke up with crunching fury. Hovercraft his own judge and jury.

O powerful Hovercraft, help us today. Cleanse our souls and point out the way. Break the ice and halt the flood. Cleanse our houses of this mud.

Ice streaming off your bow and stern You teach us lessons we never learn. Ripping, punching, tearing through, All hail to your able crew!

O terrible Hovercraft, don't let us down. Bless us and keep us so we cannot drown. Break the ice and stop this madness, End the sorrow, cure the sadness.

Fracture, rupture, destroy that glaze, Hammer and shred to earn our praise. You floating, flying, savage beast Spitting ice cubes from between your teeth.

O mighty Hovercraft, you give us pause While wreaking havoc on nature's laws. Break the ice and display your wrath To guide us on the righteous path.

We'll build you temples, mosques and ramps. We'll fuel your engines and light your lamps. We'll bed you down when day is done. We'll worship you, your ghost and your son.

O holy Hovercraft, give us the word As you swim like a fish or fly like a bird. Break the ice and blow the devil From our tawdry, ice bound level.

Billowing, streaking snow from all sides, Groaning and heaving, the ice it abides. All up and down this cursed river, Salvation is what you deliver.

Omnipotent Hovercraft, you must go away But we'll get on our knees, we'll stoop to pray. Break the ice, and to you we will sing. We beg and beseech you to return next spring.

- Herman Binkle

1.0 INTRODUCTION

This document is Volume II of a two-part report concerning ACV icebreaking operations conducted by the U.S. Coast Guard on the Illinois and Mississippi Rivers during the winter of 1977-1978. Volume I, The Executive Summary, presented a comprehensive description of key elements of the test program including an operational summary, results, conclusions and recommendations (Reference 7).

This volume contains detailed operational and engineering information on testing procedures, analysis and results. These data include findings directly related to requirements of the formal Test Plan (Reference 8) and additional information collected, which was outside the scope of the Test Plan but directly pertinent to project goals. Since the amount of data collected was so immense, it has not been possible to incorporate the entire daily operational logs or data forms from either ACV. These are, however, available as unpublished records through the U.S. Coast Guard Research and Development Center, Groton, Connecticut (References 9 and 10).

2.0 BACKGROUND

The U.S. Coast Guard is charged with providing domestic icebreaking services to satisfy the reasonable demands of commerce and to reduce the potential of flooding in the Great Lakes and on coastal and inland waterways. The severity of the 1976-1977 winter pointedly emphasized the potential consequences of a lack of icebreaking capability on the inland waterways and its subsequent impact on northern cities, industrial areas and on river commerce. Heavy river ice acutely affects the ability of towboats to transit the rivers. In ice, the number of barges pushed by tows is reduced and transit times are increased. Fuel consumption is much higher and the vessels are subject to damage from ice. The net results are delays in the delivery of needed goods and resources to northern areas, with the towboat transportation industry incurring higher costs which are eventually passed on to the consumer.

Conventional icebreakers require relatively large draft so their effectiveness is limited in shallow water. Although the Coast Guard has had vessels operating in the Mississippi River system for many years, they are primarily engaged in providing aids to navigation and are not effective in icebreaking operations because of their limited horsepower and non-ice-strengthened hulls. To overcome this deficiency several alternative measures of breaking ice were studied, and it was determined that air cushion vehicles showed promise of being effective in an icebreaking mode.

Air cushion vehicles of various configurations had been tested as icebreakers in Canada and were considered as possible candidates for the shallow or otherwise restricted waterways such as the U.S. inland waterway system. Two basic ACV configurations were employed in the Canadian tests, nonselfpropelled barges and self-propelled vehicles. Both achieved certain successful results, depending on their deployment and the ice environment.

While it was apparent that this new air cushion vehicle technology had potential for application in the Coast Guard's icebreaking programs, it was not clear what the limitations and capabilities of the vehicles were, nor what impact they would have in the areas of personnel, facilities and support services. To gain experience in some of these areas, the U.S. Coast Guard decided to conduct an Air Cushion Vehicle Icebreaker Test and Evaluation Program during the 1977-78 ice season.

Detailed plans for the project crystallized in the summer of 1977. In order to obtain operational ACV's within the short time available for the 1977-78 winter season, it was necessary to locate existing designs rather than design and build a craft for specific operational requirements. Initially, it was determined that a nonself-propelled barge type ACV could be obtained in a short time frame with greater certainty of delivery than the more complicated self-propelled type. After receiving and evaluating proposals, a barge-type ACV was leased from Mackley Ace (USA), Inc. Later, the U.S. Army agreed to make available, in a joint Army/Coast Guard effort, a LACV-30 (Lighter Air Cushion Vehicle-30-Ton Payload) self-propelled craft complete with crew and support personnel (two officers and 13 enlisted men) and support equipment.

Because the MACKACE ACV was not self-propelled, it had to be pushed through the ice. The towboat assigned this task was the CGC SUMAC (WLR-311), which offered the advantage of pushing the ACV barge with a Coast Guard vessel whose capability was known. The SUMAC is the most powerful inland

river buoy tender in the Coast Guard inventory and its hull was specially ice-strengthened for this project.

The Mackley Ace (USA), Inc. contract was awarded on 31 August 1977, and the firm subcontracted to Alberici Construction Co. of St. Louis for fabrication and assembly of the craft's components. Some of the ACV components were shipped from England, while others were purchased in the United States. The assembly of the ACV components was conducted on the bank of the Mississippi River at mile 185.6. The Coast Guard took delivery in St. Louis on 30 November 1977 and named the craft RIVER GUARDIAN.

Since the Coast Guard desired to demonstrate and evaluate the potential of ACV icebreakers, it was necessary that emphasis be placed on <u>operational</u> <u>testing</u> more than upon engineering and technical development. As a result, the Test Plan developed for the program was primarily structured to collect descriptive operational data. Some hard engineering data were obtained by incorporating engineering measurements into as many operational tests as possible.

Operational tests were conducted from 1 December 1977 to 20 March 1978. Over 450 hours of icebreaking tests were conducted using the two ACV craft during that time. The craft were used to facilitate towboat traffic by operating in the river channel. In addition, the LACV-30 successfully demonstrated its versatility by performing flood control icebreaking on a tributary river (the Kankakee).

3.0 VEHICLE DESCRIPTION

Basically, an air cushion vehicle is a craft whose weight is supported by a cushion of air which is supplied by lift fans. The air cushion is formed under the craft by a flexible rubber-coated fabric skirt system. An ACV may either be a nonself-propelled lift platform, i.e., barge-type, or it may have an active propulsion system and thus be self-propelled.

The RIVER GUARDIAN was a nonself-propelled steel barge-type ACV, simply designed and built according to conventional boat construction techniques. Lift was obtained from two standard-type diesel engines which drove two centrifugal fans. The speed of the RIVER GUARDIAN was limited to that of its pusher vessel. Figures 3-la and 3-lb give the general characteristics of the RIVER GUARDIAN.

The LACV-30 (Lighter, Amphibious, Air Cushion 30-Ton Payload) is a fully amphibious, high speed craft, designed for transporting military cargo payloads in the 25- to 30-ton capacity. The LACV-30 is powered by two twin-pack gas turbine engines rated at 1800 maximum shaft horsepower each. Lift capability is provided by two 7-foot centrifugal, 12-blace, fixed pitch fans. Propulsion capability is provided by two 9-foot, 3-blade variable pitch propellers, and it can attain speeds in excess of 40 knots. It is constructed of hollow core marine aluminum using aerospace technology. Figure 3-2 gives the general characteristics of the LACV-30.

The U.S. Coast Guard Cutter SUMAC (WLR-311) stationed in the Second District is the largest Coast Guard inland river buoy tender. For most river operations, the vessel is capable of operating at speeds up to 10 mph. However, its speed is drastically reduced in ice-clogged waters. For its role in pushing the RIVER GUARDIAN, the SUMAC's hull was modified by the addition of ice deflectors to protect the keel coolers, rudders and propellers. Figure 3-3 gives the general characteristics of the SUMAC.

GENERAL CHARACTERISTICS

- 45 FT. x 55 FT. • GENERAL DIMENSIONS
- 600 HP EACH (MODEL 16 V 71-N, 2109 RPM) TWO GENERAL HOTORS DIESEL ENGINES
- FLOW RATE : 4800 CFM EACH(1200 B.A. SISW, DESIGN RPM = 2000) . TWO ALLDAY & PEACOCK CENTRIFUGAL FAHS SINGLE INLET,
- . TWO 1000 GALLON FUEL TANKS
- . HOVER HEIGHT BETWEEN 4 FT. 8 5 FT.
- 0.95 PSI & 1.3 PSI (DEPENDING ON BALLAST) . CUSHION PRESSURE MAINTAINED BETWEEN

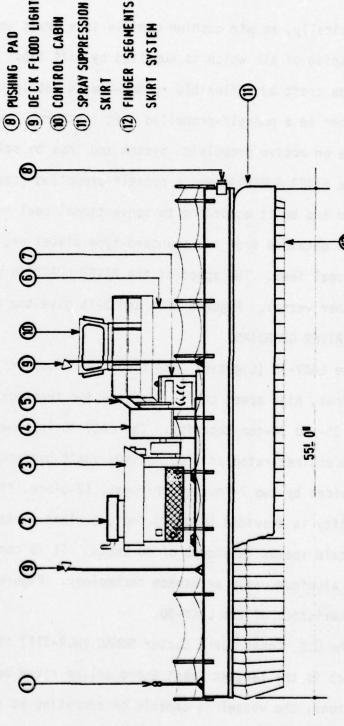


Figure 3-1a. River Guardian Profile, Side B-C (NOTE : SPRAY SKIRT CUT BACK TO REVEAL SEGMENTS)

IDENTIFICATION TABLE

- (1) BOW RUNNING LIGHT
 - (2) ENGINE #1 EXHAUST
 - 3 ENGINE #1
- G FAN#1
- (5) STEPS TO OBSERVATION DECK & CONTROL CABIN
- (6) STORAGE AREA
- 7) GENERATOR
- (B) PUSHING PAD
- (9) DECK FLOOD LIGHTS
 - (10) CONTROL CABIN
- (12) FINGER SEGMENTS OF SKIRT SYSTEM

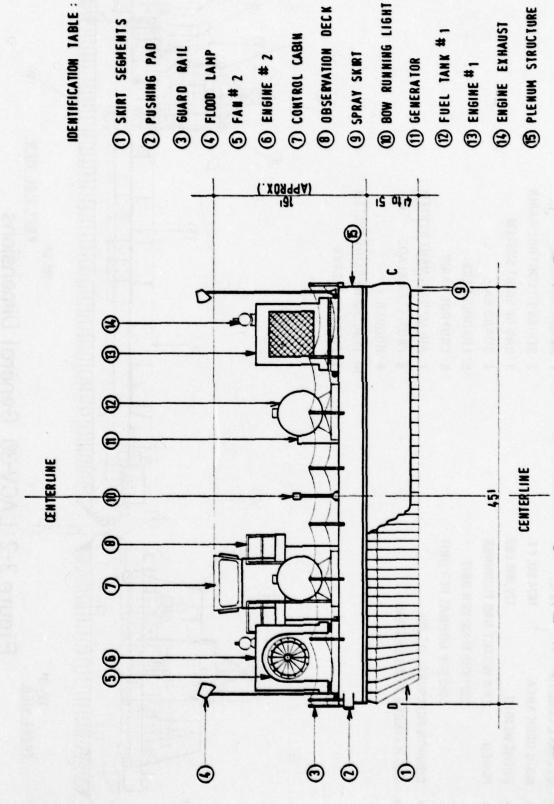


Figure 3-1b. River Guardian Profile, Side C-D (NOTE : SPRAY SKIRT CUT BACK TO REVEAL SEGMENTS.)

GENERAL CHARACTERISTICS

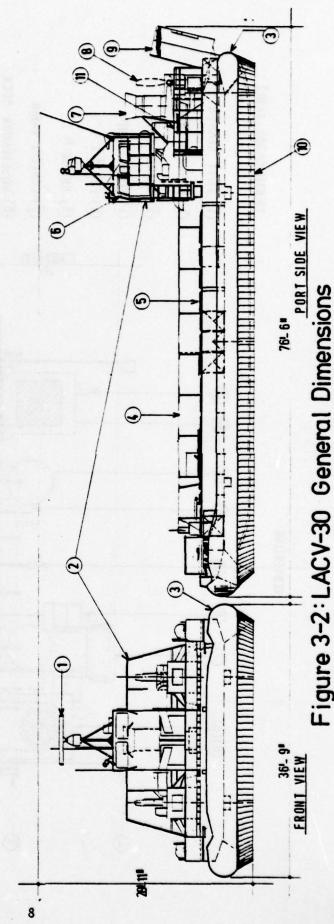
- .9.91 OVERALL LENGTH
 - **MAIN DECK AREA**
- 1674 SQ. FT.
 - 125,000 LBS. **2 P & W STGT GAS TURBINES GROSS WEIGHT** POWER
- 1400 SHP NORMAL PER UNIT **1800 SHP MAX. PER UNIT**
- **CUSHION PRESSURE 0.5 PSI**
- **BAG & FINGER SKIRT SYSTEM**

IDENTIFICATION TABLE:

- RADAR ANTENNA
- STAIRS TO CONTROL CABIN
- **BAG OF SKIRT SYSTEM**
- **GUARD RAIL**
- **LOADING DECK**

2 9

- CONTROL CABIN
- **AIR MANAGEMENT SYSTEM** 8 PROPELLER BLADE
- RUDDER 6
- 10 FINGERS OF SKIRT SYSTEM
- 11 LIFT FAN INTAKE



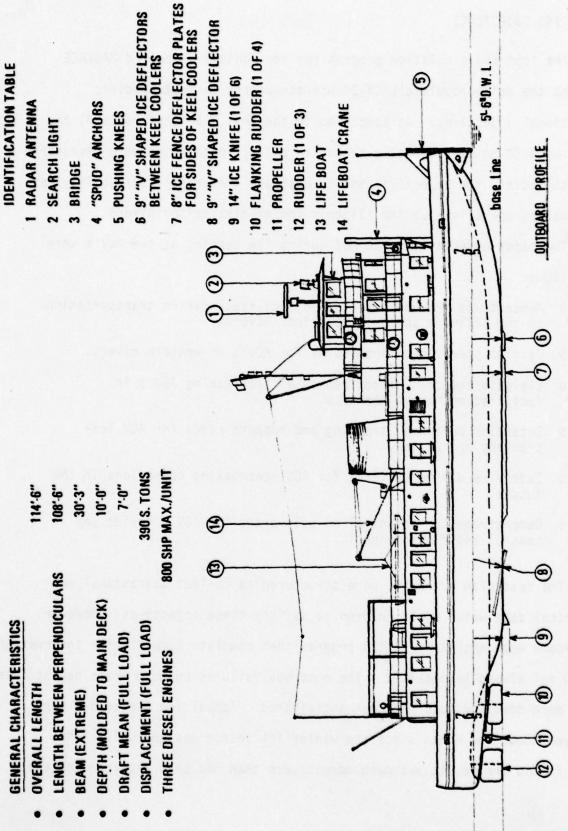


Figure 3-3: United States Coast Guard Cutter-Sumac

4.0 TEST OBJECTIVES

The test and evaulation program for the nonself-propelled MACKACE ACV and the self-propelled LACV-30 was structured to meet specific operational objectives. As specified in the Test Plan (Reference 8) the ACV's were to be operationally tested to collect data which were necessary to both satisfy the objectives and to evaluate the craft's suitability for icebreaking operations on the Illinois and/or Mississippi Rivers.

The major objectives to be met during the testing of the ACV's were as follows:

- Demonstrate the use of ACV's to facilitate marine transportation on the Illinois and/or Mississippi Rivers.
- o Develop operational procedures for ACV's on western rivers.
- Evaluate the cost effectiveness of icebreaking ACV's in facilitating marine commerce.
- Determine personnel training and support needs for ACV icebreaking operations.
- Determine design criteria for ACV icebreaking operations in the future.
- Demonstrate the potential of self-propelled ACV's in ice jam control and flood control.

The test plan scenarios were structured to collect operational and technical data which would be used to satisfy these objectives. However, it became apparent early in the program that complete quantitative information could not always be obtained. The numerous failures on both craft necessitated much more downtime than had been anticipated. Actual ice operations were delayed about one month since the winter ice season was unusually late.

When it did arrive, it was much more severe than had been expected and the

nature of the ice to be broken was much more complex than had been predicted. These heavy ice conditions affected the width and depth of the channel to which the SUMAC was restricted because of its size. In addition, conditions varied so widely from day to day that it was never possible to run identical tests on successive days with the same environmental parameters. Yet another problem was arranging for the availability of cranes and other support facilities to meet tight operational schedules. All of these factors combined to force modification or postponement of certain tests and, therefore, information necessary to meet the objectives is more complete for some than it is for others.

A Steering Committee was formed which met periodically in Peoria to redirect the priorities of the individual tests within the overall objectives of the test plan. In the end, little information was actually acquired on the cost effectiveness objective. Also, whereas training and support needs and design criteria were not addressed in depth, sufficient data was collected and subjective observations were made by experienced investigators to permit valid conclusions to be drawn and recommendations to be made.

5.0 OPERATIONAL SUMMARY

5.1 General

This section describes the operational aspects of the Test and Evaluation program, specifically identifying what data were collected and how they were collected. In addition, it presents a narrative account of the river ice environment and conditions encountered by the two ACV craft as they performed icebreaking tests.

5.2 Data Acquisition

A set of fourteen (14) specific tests were developed in the Test Plan. They are listed in Table 5-1. Individual test descriptions were initially developed for the nonself-propelled ACV and subsequently adapted to the self-propelled LACV-30. Nine data collection forms were used in collecting and recording the data during each test. A Test Integration Form was prepared for each test and was used to record each test parameter relative to the test time and river location baseline. Table 5-2 lists the data collection forms and test team members responsible for recording data collected during each test.

At the outset of the project, it became evident that there might be a problem in recording and correlating the data collected at vious locations on the SUMAC and its ACV bow. To ensure proper data correlation and to establish a time and location baseline for test data integration, the testing operations were directed from the bridge of the SUMAC by the test director and/or test engineer. From this location, all the data gathering positions were coordinated by the use of the boat's communications system.

TEST NO.	TEST NAME	TEST PURPOSE
1-91	Measure Commerce	To measure the elementary components of river commerce.
1-92	Towboat Transit Measures	To determine the effect of air cushion vehicle icebreaking upon commercial towboat fuel consumption and transit times in ice.
1-03	Towboat Maneuvering Measures	To determine the effect of air cushion vehicle icebreaking upon commercial tomboat maneuvering in ice.
1-64	Icebreaking Techniques	To learn the capabilities of the air cushion vehicle icebreaker to break ice under various operational conditions and constraints and to develop appropriate techniques for unique applications.
1-05	Ice Pattern Documentation	To determine size and shape of the ice when broken by the air cushion vehicle ice- breaker and determine the clearance (i.e., the width) of the track of broken ice.
1-96	Passing Maneuver	To determine the characteristics of the air cushion vehicle icebreaker when passing a towboat.
1.97	Stopping Maneuver	To study the relation between ice thickness and the stopping distance of the air cushion vehicle icebreaker following engine shut-down.
1-08	Cost Effectiveness	To determine rate at which fuel is consumed and ice is broken for different combina- tions of ice thickness and speed.
T- 9 9	Impact on Coast Guard	To identify the needs for additional training and the types of personnel necessary to support an air cushion vehicle in the Coast Guard. Also to identify the additional hardware and/or support services necessary to maintain an air cushion vehicle.
T- 0 10	Design Criteria	To determine the technical aspects of air cushion vehicle design. This as a matter of course must include the performance of the pusher vessel also.
1-011	Open Water Characteristics	To obtain engineering and operational information about the air cushion vehicle icebreaker when operating in open water.
1-012	Turning Maneuver	To document the characteristics of the air cushion vehicle icebreaker in a standard turn in open water and ice.
T-013	Icebreaking Maneuver	To observe and photograph ice breaking operations and study the physical mechanism by which the air cushion vehicle breaks ice.
1-014	Ballasting	To trim and ballast the air cushion vehicle to desired values of cushion pressure and platform level.

Table 5-1 Fourteen Tests from the Test Plan

FORM	TITLE Test Description	RESPONSIBLE PARTY
F00	Test Integration Form	Test Coordinator (Support Engineer)
F01	Atmospheric Conditions	Support Engineer
F02	Ice and River Conditions	Support Engineer
F03	Operations	Support Engineer
F04	Engineering, Pusher	Support Engineer
F05	Engineering, ACV Bow	MACKACE Personnel
F06*	Maintenance, Support and Casualty	Support Engineer
F07*	Casualty Report	Support Engineer
F08*	Commerce	Support Engineer
NA	Ice Classification Sheets	Support Engineer
NA	Finger Segment Damage Diagrams	Support Engineer

^{*}These forms were actually never used during the tests. Supporting contractor engineers recorded maintenance and damage reports separately as necessary. The finger segment failures were recorded on separately developed diagram sheets and commerce information was merely recorded informally as it was available.

Table 5-2 Test Form Responsibility

The locations coordinated and the data collected for each were: SUMAC:

- o Deck ice and weather conditions
- o Engine room horsepower, fuel consumption, engine RPM
- Bridge time, location, speed, ACV cushion pressure, trim, maneuver type

RIVER GUARDIAN (ACV Bow):

- o Control cabin trim, pitch, engine RPM, cushion pressure
- o Deck ice conditions, hover height, icebreaking effects

 From the bridge, the test director and test engineer established the data record by signifying at what times individual data items should be recorded. Thus, a data baseline relative to time, location, and ice environment was maintained.

Data gathering procedure on the LACV-30 was quite difficult due to space limitations and the fact that it was not possible to use the deck when the craft was underway because of heavy spray. Data were, therefore, recorded mostly from the cabin by the test director and engineer. This meant that measurements of ice conditions were actually estimates, except when the vessel was stopped and readings were taken from the deck.

5.3 Operations and Testing

In all, more than 57 tests were conducted and documented on the SUMAC/ACV and the LACV-30. Table 5-3 summarizes the tests conducted by the SUMAC/ACV. It should be noted, however, that many of the tests consisted of multiple runs and data points. Thirty-five multiple run tests were conducted which resulted in a total of 106 data items. The raw data for each test run are contained in Reference 9. An example of a data summary sheet is shown in Table 5-4a, b, c, d, and e.

	TEST RUN TYPE	NO. OF TESTS	NO. OF RUNS			
roı	Measure Commerce	1	1			
02 03	Towboat Transit Measures Towboat Towboat Maneuvering Measures Transi		7			
704	Icebreaking Techniques	8	38			
05	Ice Pattern Documentation	See Ice Cla	assification			
06	Passing Maneuver	1	1			
07	Stopping Maneuver	2	9			
08 09	Cost Effectiveness Impact on Coast Guard	Continually	y Monitored			
10	Design Criteria	10	35			
111	Open Water Characteristics	1	8			
12	Turning Maneuver	4	4			
13	Icebreaking Mechanism	See Ice Classification (Handled by Ice Classification System)				
114	Ballasting	3	3			
OTAI	*	35	106			

^{*} There were 35 formal tests conducted in accordance with the Test Plan. However, many tests included multiple runs, hence there were 106 runs in 35 tests.

Table 5-3 Summary of Tests Conducted by the SUMAC/ACV

-:		0	0	0	0	*	5.	13		1	Ě	1	16.	16*
	COMMENTS	Down bound travel	Up bound travel	Down bound travel	Down bound travel	Mostly pushing the ice, some	Test measurements on ice incomplete	Tested on keel	Up bound-only	surements were taken. Other engine values assumed the same	This test was run on keel engine only	Down bound	Up bound	Up bound
	ICE CONDITIONS	No ice, open water characteristics	RIM WITHOUT ACV BOW Unconsolidated brash			100	Nerrozen brasn up to 14 inches thick, avg. 6 to 8 inches consolidated	New solid ice	Ice had been previously broken	Previously broken but consolidated ice 214 in. thick, ice chunks 10 to 15 ft. across knitted together with 4 to 6 in. ice, acted as large flow	Same as previous entry			
	AVG FUEL GPH		BOW	BOW	CV BOW	CV BOW	30	30		30	30	30	30	30
,	PRES. PSI	RUN MITHOUT ACV BOW	RUN YLTHOUT ACV	WITHOUT ACV	WITHOUT ACV BOW	THOUT A	0.94	1.05		1.05	Y.	0.97	1.04	1.04
ACV	н. Р.	RUN WI	RUN W	S.	RUN W		!	1		ī	ž	1	1	-
	RPM	TEST	TEST	TEST	TEST	TEST	1600	1800		1800	ž	1900	1800	1800
	VEL.	8.9	80	9.1	10.4	3.4	2	2.5		2.8	6	Ş	3.2	3.5
ŗ	FUEL	34.4	33.9	46.3	84.2	11.7	19	6		98.1	35.4	7.3	69.4	8.79
SUMAC	AVG	900	006	1000	1200	1044	1000	124		1132	1140	423	964	1003
	TOTAL H.P.	580	579	793	1381	1067	1015	147		1674	595	128	1223	1148
	RUN	-	2	-	5	-	-	-		~	-	-	-	2
	T.C. NO.	100	100	100	100	003	900	900		900	900	000	800	800
	DATE	1977	12/29	12/29	12/29	12/29	12/30	1978		1711	11/1	1/11	1/14	1/14

• Estimated
•• Relative Strength Factor

Data Summary Sheet for Tests Conducted by the SUMAC/ACV Table 5-4a

-:		12.		12*	-		12*	16	16	18	2	14	15	15
_=	CONFIENTS	Up bound 1	in the	up bound, ice previously broken by tows	This ice appeared to be unconsoli- dated with only	20		Up bound, channel 1 broken 70 ft. vide		Up bound, egannel 1 broken 70 ft wide	-	-	Started into ice but stopped in 1 1's boat lengths	Started from dead in water bow into 1 ice
	ICE CONDITIONS	Refrozen brash chunks 8 to 12 inches thick, 4 to 10 ft. wide knitted with 2 in. of new ice	Data taken in channel narrov betveen lakes.		Brash ice plates 4 to 10 ft. across 4 to 8 in.		sh chunks es thick, 4 de knitted f new ice	Ice was solid covered	Ice was solid covered	Ice was solid covered	This ice was very thick slush and hard to classify	Very thick packed ice		
	AVG FUEL GPII	BO	BOM	208	1		80	30	2	30	30	30	BON	NO.
^	PRES.	TEST RUN MITHOUT ACY BOW	RUN WITHOUT ACY	TEST RUN MITHOUT ACY BOM	A Trouble and Post		TEST, RUN WITHOUT ACY BON	1.06	1.06	1.10	1.03	0.91	TEST RUN WITHOUT ACV	TEST RIN MITHOUT ACV BON
ACV	н.Р.	RUN WI	RUN WI	RUN WI			RIN WI	1	1	1	1	1	IN NI	TIN NIL
	RPH	TEST	TEST	TEST	-	2	TEST	1800	1800	1800	1750	1750	TEST	TEST
	VEL.	4.2	6.9	6.2	, ,	:	4.7	1.8	1.6	3.8	3.4	2.7	¥	5
31	FUEL	111.7	105.3	97.0	3		106.0	15.5	15.4	34.5	0.49	66.2	18.8	110.8
SUNANC	AVG	1154	1097	1066	11.36	1	1046	294	280	806	686	686	633	1180
	TOTAL H.P.	1746	1895	1671	7181		1818	292	292	839	1100	1104	319	1882
	NG.	-	4	5	,	•	·	-	~	3	7	-	-	7
	T.C. NO.	600	600	600	8	3	60	010	010	010	100	110	012	012
	DATE	1978	1111	71/1			1/11	17.51	1/31	1/21	1/13	1/23	1/24	1/24

* Estimated ** Relative Strength Factor

Data Summary Sheet for Tests Conducted by the SUMAC/ACV Table 5-4b

~	S. F.	2	71	91	9	4	*	2	2	=	12	13	П		T
	COMMENTS	Data taken after	Data taken from a running start	Came to a full stop									Down bound pre- vious broke run 15	Down bound pre- vious broke run	Down bound pre- vious broke run
	ICE CONDITIONS			Rafted up over 5 ft. thick	Unconsolidated brash 1 to 2 feet thick	Unconsolidated brash I foot thick	Unconsolidated brash 2 ft., imbedded chunks 10 ft.	Brash unconsolidated 2 ft. to 3 ft. thick	Brash unconsolidated 2 ft. to 3 ft. thick	Brash unconsolidated 2 ft. to 3 ft. thick imbedded chunks	Brash unconsolidated 2 ft. to 3 ft. thick imbedded chunks	Brash unconsolidated 2 ft. to 3 ft. thick ice balls 8 ft. thick			
	AVG	V BOW	/ BOW	V BOW	30	30	30	30	30	30	92	30	30	30	30
	PRES.	RUN WITHOUT ACV	MITHOUT ACV	WETHOUT ACV	96.0	96.0	96.0	66.0	0.99	0.99	0.99	96.0	0.99	0.99	1.00
ACV	H.P.	RUN WIT	RIM WIT	RUN WE	NA.	4N	ž	×	NA	¥	\$	ž	¥	1 5	ž
	КАМ	TEST	TEST	TEST	1650	1650	1650	1750	1750	1750	1750	1750	1800	1800	1800
	VEL.	2.9		¥	3.8	3.4	2.5	3.6	3.7	3.5	3.1	3.4	3.7	3.4	3.5
,	FUEL	115.5	18.3	112.8*	14.5	13.8	16.6	14.2	11.4	14.2	50.3	30.4	20.2	12.6	11.8
SUPAY.	AVG	1153	1119	1115	588	579	109	185	165	185	921	783	676	195	541
	TOTAL H.P.	1955	077	1805	244	240	287	241	194	239	857	\$21	345	221	201
	M	3	•	5	-	7	-	-	7	•	4	~	•	-	80
	T.C.	012	012	012	013	013	613	710	910	910	910	910	110	710	710
	DATE	1978	1/24	1/24	2/3	2/3	2/3	2/4	2/4	2/4	2/4	2/4	2/4	2/4	7/7

* Estimated ** Relative Strength Factor

Data Summary Sheets for Tests Conducted by the SUMAC/ACV Table 5-4c

~	. F.	13	14	13		15			20	23		=	=	20*	10	10	10	10	101	10	=
	COPPLEXIS																	Pressure gauge not working	Pressure gauge not working	Pressure tap frozen	and the effects of ion were observed. ed to accumulate.
AND THE PERSON OF	ICE CONDITIONS	Plate ice broken into 3 ft. chunks	Refrozen brash, Consolidated	Consolidated brash & 4	Consolidated brash & 4 Inches thick	Consolidated brash, unf- form thickness % 5 in.			This ice was very thick	Uniform icel3 - 15 in. thick		Unconsolidated ice slush and rubble		Rubble 20 in. thick, 12 - 14 ft. across						Consolidated brash	Several speeds were run and the effects of ice build up in the cushion were observed. At slower speed ice seemed to accumulate.
	AVG FVEL GPH	30	30	30	30	30	30	30		30	30	30	30	30	BOW	BOW	BOW	30	30	30	30
	PRES. PSI	NA	VN.	10.1	1.03	1.04	0.97	1.05	DED	NA	¥	1.07	1.07	1.07	PLIN WITHOUT ACY	RUN MITHOUT ACY	RIJN WITHOUT ACY	1	1	1	1
ACV	H.P.	MA	NA	NA	NA	¥	NA	NA	UNRECORDED	N.	¥	NA	NA	NA NA	IN MI	RUN WIT	IN MI	¥	Y.	¥.	¥.
	K-Ja	1700	1700	1700	1700	1700	1700	1900	5	1800	1800	1800	1800	1800	TEST	TEST	TEST	1850	1850	1850	1700
	VEL.	2.4	1.6	3.6	2.6	3.1	4.1	15.7	¥	4.5	3.1	5.5	5.0	5.4	6.5	7.8	9.6	13.6	4.8	9.6	3.4
21	FUEL	14.8	16.6	39.2	18.6	28.8	52.9	52.9	541 11.7	91.6	1057 94.5	33.6	785 29.9	85.4	0.09	26.7	922 56.2	52.4	28.0	48.0	55.1
SUMAC	AVG	579	599	851	179	159	941	932		1050		815		1089	897	920		976	871	913	927
	TOTAL. H.P.	255	286	585	316	200	903	806	197	1528	1682	295	503	1488	1062	980	696	917	11.5	808	920
	RUN	1	2	3	7	2	9	1		-	2	3	4	2	-	2	3	7	5	9	-
	T.C. NO.	910	910	910	910	910	910	910	910	710	017	110	017	710	023	023	023	023	023	023	024
	DATE	1978	3/6	3/6	3/6	3/6	3/6	5/6	9/2	1/1	1/1	1/12	1/2	1/1	2/10	2/10	2/10	2/10	2/10	2/10	2/11

* Estimated ** Relative Strength Factor

Data Summary Sheets for Tests Conducted by the SUMAC/ACV Table 5-4d

-	F.	13	=	13	13	14	14	
	COMMENTS	Pressure guage not working on ACV			•		•	chan wendy specific data (Censimalive) chan cest cata were collected and respire conference to a committy of testal comp
	ICE CONDITIONS	See T.C. No. 024, run 1	See T.C. No. 024, run 1	See T.C. No. 024, run 1	See T.C. No. 024, run 1	See T.C. No. 024, run 1	1 1	in the second of the second second of the se
	AVG EWEL	30	30	30	30	30	30	tion Keedo laas and market and and are
	PRES.	1	1	:	1	1	:	na 87EJ kwawaka 81 na binish darah wat dal
ACV	H.P.	¥	¥	NA NA	¥	KN	NA.	De March 1979 Noen testing Work Ster Forth St
	M M	1700	1700	1700	1700	1700	1700	
	VEL.	4.7	4.9	4.3	3.8	4.7	4.6	Go inc docamina they have no surrous and
SUMAC	FUEL	97.8	54.0	54.3	53.8	9.76	9.06	ander for each one arease for yet
	AVG	1105	937	937	176	1118	1109	wadana et Theore coarte de conseque
	TOTAL H.P.	1607	925	925	918	7651	1559	AND THE MOTOR SOUR AND THE STATE AND THE
	3		3	7	5	1	1	
	T.C. NO.	024	720	024	700	700	024	
	DATE		111/2	111/2	+	+	2/11/2	MINERAL MANAGEMENT OF THE PROPERTY OF THE PROP

.. Relative Strength Factor

Data Summary Sheet for Tests Conducted by the SUMAC/ACV Table 5-4e

The state of the s

Tests with the LACV-30 were conducted with procedures similar to those used with the SUMAC/ACV; however, because of the great differences in the characteristics of the two craft, operational techniques required certain modifications. More than twenty specific data items relative to the Test Plan plus numerous other test data were collected and recorded in the Daily Operational Log (Reference 10). A summary of tests completed by the LACV-30 is presented in Table 5-5.

The SUMAC/ACV and LACV-30 were both available for testing in the Peoria area as scheduled. The SUMAC/ACV combination was in the area from 16 December 1977 to 8 March 1978. In addition, the four-day transit from St. Louis, Missouri, to Peoria, Illinois, can be considered in retrospect as test time since much valuable information was obtained. The LACV-30 reported for duty with the Coast Guard on 15 January 1978 and was in operational status until 18 March 1978 when testing was concluded. Tests were conducted to obtain operational and engineering data. Also nontesting time necessary for repairs and preventive maintenance provided an opportunity to collect additional data relative to the craft's impact on the Coast Guard design criteria for ACV icebreakers and cost data for icebreaking operations.

A breakdown of activities for each craft is presented in Table 5-6. In order to compile the table, each day's operation was classified according to the activity which predominated. For example, if five hours of one day were devoted to operations and three hours to repairs, the whole day was considered as "operations".

It can be seen that both vehicles had extensive periods of downtime for maintenance and repairs. The RIVER GUARDIAN had few mechanical problems

	TEST	TIMES PERFORMED	
SPT-1*	Towboat Transit Measures	4	
SPT-2	Towboat Maneuvering Measures	10	
SPT-3	Icebreaking Techniques	Continuous Experimentat	ion
SPT-4	Ice Pattern Documentation	Continuously Observed	
SPT-5	Overtaking Maneuver	Numerous	
SPT-6	Stopping Maneuver	Numerous, 1 Emergency	
SPT-7	Cost Effectiveness	1	
SPT-8	Design Criteria	Documentation as Applic	abl
SPT-9	Open Water Characteristics	2	
SPT-10	Turning Maneuver	Numerous	
SPT-11	Icebreaking Mechanism	Continuously Observed	
SPT-12	Abandoned Barge	1	
SPT-13	Sunken Barge	1	
SPT-14	Impact Upon Coast Guard	Continuously Monitored	
ADDITIO	NAL TESTS AND OPERATIONS		
	Extended Deployments (Up to 160 mi. radius)	4 3017 99-3	
	Multi-Day Operations (away from base	at ANFAC) 21 Days	
	Breaking Out of Lock Gates	TWISTER LOOP TO CONSIDER	
	Ice and Channel Profile Survey	3	

^{*} SPT: Self-propelled Craft Test

Table 5-5 Summary of Tests Conducted by the LACV-30

SUMAC/ACV			
	Operations (Days)	Maintenance and Repair (Days)	Other (Days)
12 - 31 Dec	6	0	14
1 - 31 Jan	4	16	11
1 - 28 Feb	8	18	2
1 - 8 Mar	3 200000	1	4
Totals	21	35	31
LACV-30			500000000000000000000000000000000000000
	Operations (Days)	Maintenance and Repair (Days)	Other (Days)
15 - 31 Jan	1	14 *	2
1 - 28 Feb	14	7	7
1 - 18 Mar	12	6	0
Totals	27	27	9

^{*} Including Set-Up Time

Table 5-6 Breakdown of Activities for SUMAC/ACV and LACV-30

but required frequent repairs to the skirt system because of ice damage.

The LACV=30, on the other hand, was seldom down for protracted periods of time for skirt repair but needed a great deal of preventive maintenance to assure effective operation of its comparatively complicated propulsion system. Detailed records of maintenance and repairs conducted on each craft are provided in References 9 and 10.

The various tests conducted with the SUMAC/ACV and the LACV-30 not only provided data and information to support the objectives but also provided engineering and operational data which advanced the state-of-the-art in ice classification techniques, icebreaking effectiveness measurements, ice management techniques, operational noise problems and skirt failure documentation. A more complete discussion of these items is given in Section 6 of this report.

5.4 Flood Control Operations on the Kankakee River

In addition to icebreaking operations to facilitate towboat traffic, the LACV-30 was deployed to the Kankakee River to evaluate its potential in flood control operations. On the Kankakee, where commercial towboat traffic does not transit, there is no consistent channel depth and, once ice forms, the river is frozen until the end of the winter season. As the season changes from winter to spring, with warm days and freezing nights, the ice begins to break up. Variations in temperature cause the ice to begin breaking up. Ice chunks break off at expansion crack joints, which develop due to the thaw-freeze cycles, and these chunks begin to flow downstream. Channel characteristics, however, influence their path. At some points along the rivers

and streams, the channel widens but depth decreases, and ice chunks, which can be as thick as 18 to 20 inches, ground in these shallow areas and eventually pile up to a point that they dam the river. Damming can also occur at river bends and bridges. As temperatures continue to rise, the watershed increases upstream of these dams, backing up the river, overflowing its banks, and flooding the adjacent lowlands.

Because of its amphibious capability and ability to traverse obstructions, the self-propelled ACV has a good potential for success in ice jam and flood control operations. The LACV-30 was so operated in the lower Kankakee River, just above the Dresden Island Lock and Dam (110 miles north of its Peoria, Illinois base) from 1 to 18 March 1977 with good results.

5.5 River Ice Environment

To fully document the river ice environment, an overall ice characterization of the test area on Peoria Lake and the Illinois River was conducted. At the outset of the project, little ice formed on the river. Operations during December were for the most part conducted on open water or in water with only about 40 percent of the channel covered with thin ice. By early January, however, ice began to accumulate. During the months of January and February, the lake was mostly covered with uniform ice except in the channel area where barge traffic traveled. In the channel, there was a mixture of rafted ice, slush, brash, and imbedded plates. Figures 5-la, b, and c summarize representative samples of ice conditions taken by aerial and surface surveys. At various times during the two months, the ice accumulated and thickened, but overall it stayed relatively stable throughout the test period.

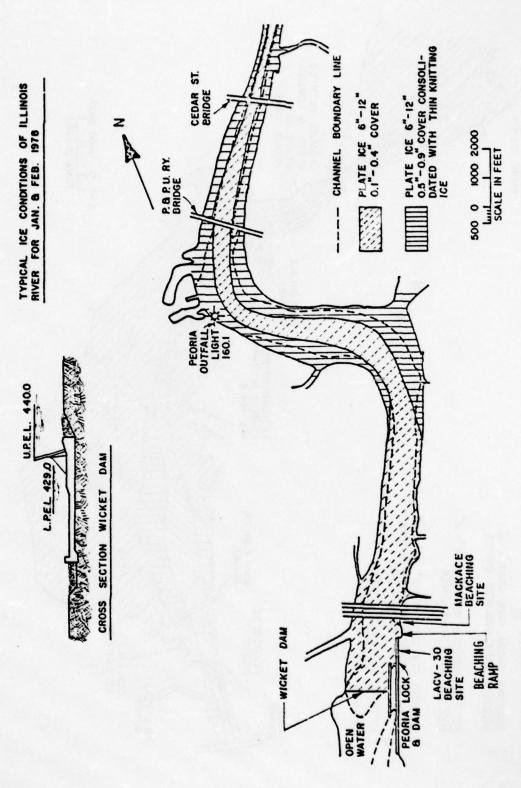


Figure 5-la Illinois River Ice Conditions

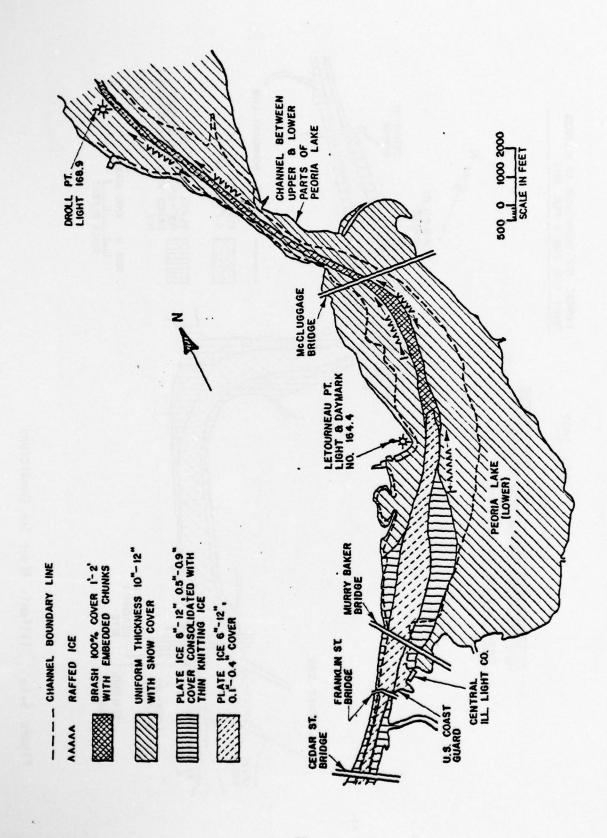


Figure 5-1b Illinois River Ice Conditions

The second second second second second

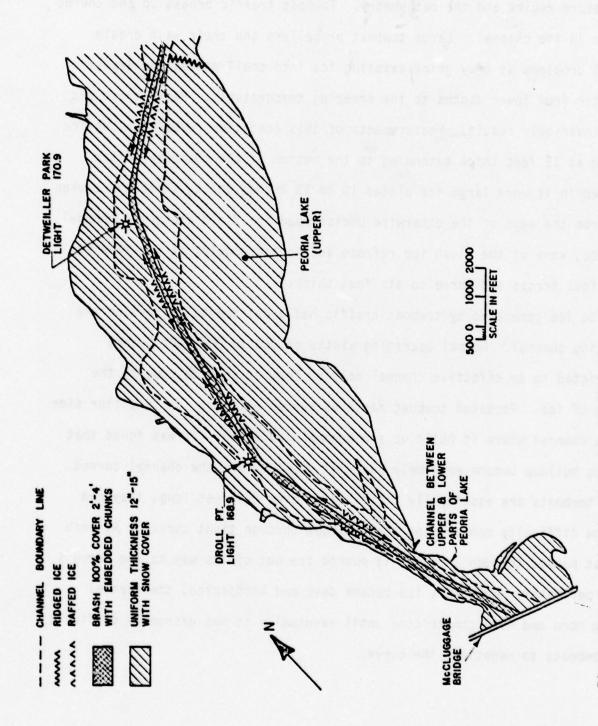


Figure 5-1c Illinois River Ice Conditions

The significant characteristics of river ice are directly related to the action of towboats passing through it, the flow of the river, the temperature regime and the bathymetry. Towboat traffic breaks up and churns the ice in the channel. Large towboat propellers and their wash create special problems as they grind existing ice into small pieces and expose new water from lower depths to the freezing temperatures. Heavy brash and slush invariably result. Measurements of this condition revealed slush ice as much as 15 feet thick extending to the bottom of the channel. Often embedded in it were large ice plates 10 to 15 inches thick, which had broken away from the edge of the otherwise undisturbed ice field along the channel. At times, some of the slush ice refroze and formed modules of ice five to eight feet across and three to six feet thick.

The ice generated by towboat traffic had a detrimental effect on the operating channel. Normal operating widths of 200 to 300 feet became constricted to an effective channel only 60 feet or so wide, due to the growth of ice. Repeated towboat passages pushed existing ice to either side of the channel where it built up and became hardpacked. It was found that the ice buildup became extremely heavy at points where the channel curved. Since towboats are essentailly rigid bodies over 200 feet long, they had extreme difficulty maneuvering their barges through tight curves. As each towboat passed through a curve, it pushed ice out of its way to the curve's outer perimeter. When this ice became deep and hardpacked, the channel became more and more constricted until eventually it was extremely difficult for towboats to negotiate the curve.

To better understand the ice environment in the Peoria Lake channel, an ice profile survey was conducted with the LACV-30. One of the measured profiles is shown in Figure 5-2. It can be seen that in the shallow portions of the river, the ice was plate ice, only 15 inches or so thick, but it had become up to 15 feet thick with slush ice in the main channel due to the action of towboat traffic. The profile was taken at an end point of a turn and indicates the growth of ice on the outer (left) edge of the turn. It can be seen that, although the channel depth allows some 350 feet of width for operations, the effective operating channel has been reduced to about 150 feet at the inside of the turn. Towboat traffic had packed the ice up so tightly that it extended above the surface as much as four feet high in certain areas.

In addition to ice profile measurements, various measurements of ice thickness, temperature, ice types and the behavior of broken ice were recorded periodically during each day of testing. Ice borings were made at locations where icebreaking tests were conducted. Table 5-7 shows some of the results of those samples. To meet the needs of the test program, comparisons of day-to-day and test-to-test data were necessary. A system of translating ice measurement values into ice classification terms was developed by characterizing the river ice environment in terms of a Relative Strength Factor (RSF) which allowed direct comparison of test data. The rationale for and use of the RSF are presented in Section 6 of this report.

The operation of the LACV-30 on the Kankakee River for flood control required environmental documentation for that location also. The ice environment was similar to the Illinois River, except that the Kankakee River does not have barge traffic. Ice profile and in situ measurements were made at

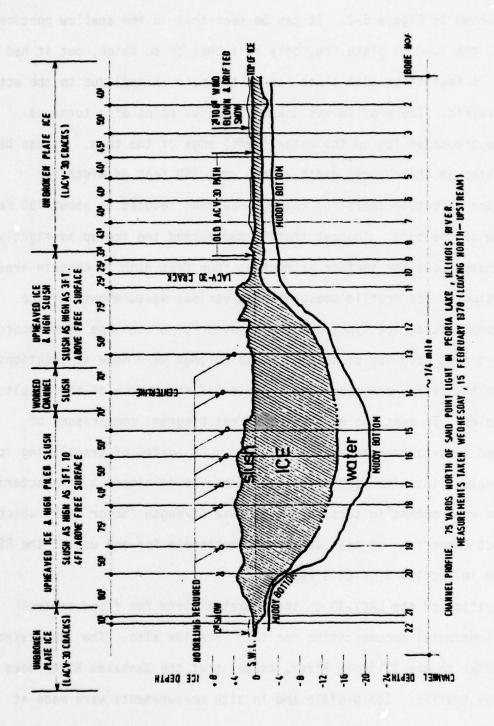


Figure 5-2 Ice and Channel Profile

pres in						0.00		Г					
no Langon en	RSF	16	=	14	91	80	10	13	50	13	14	10	13
on one of the second of the se	ICE DESCRIPTION & COMMENTS	New plate ice very little knitting and closely packed	Brash ice next to previously broken channel	Ran into heavy ridge of ice piled up to tow boats	Snow cover 3 inches. Ice is very thick.	ice was consolidates with	Brash ice in place pushed up 2 to 4 feet thick	Plate ice refrozen into a uniform field	This ice was very thick previously unbroken lake ice.	Brash ice in channel with imbedded plate ice 8 to 10 in.	In channel ice 4 to 6 in. with	No comments available	Refrozen brash & rubble field
	FIELD TYPE	Uniform Thickness-Cold	2 Brash-melting	2 Brash-melting	3 Uniform Thickness-Cold	Unconsolidated Brash	O Unconsolidated Brash	2 Uniform Thickness-Cold	1 10 Uniform Thickness-Cold	2 Brash-cold	6 Brash-melting	1 Brash-melting	2 Brash-cold
ting Thickness (in)		9	2	2		-	0	1	10	2		1	1
	2814	-	7	-	5	12	6 10 12	2		~	2	219	9
th) exic nisto to e	7819	6	m	00	12 10	6 10 12	10	2 10	16 25	4	4		7
e Thickness (in)	1819	9	7	10	12	9	9	5	16	3	80	4	12
	 80.	010	160	110	012	013	10	910	110	110	024	023	028
	LOCATION	Channel 164.4 Mi.	Channel 164.4 MI.	Channel 165.3 Mt.	Lake 164 Mi.	Channel 159.9 Mkr.	Channel 167 Mi.	Channel 166 Mi.	Lake 172 Mi.	Channel 167.4 Mi.	Channel 167 Mi.	Channel 167 Mi.	Channel
	MTE	_	1/23/78		8	-	-	2/6/78	-	2/1/18	-		2/21/78

* I.C.N. = Test Control Mumber and is used to correlate the handling of data collected during a particular test.

** RSF = Relative Strength Factor

Table 5-7 Summary of Illinois River Ice Conditions

various locations along the test areas. Overall, the ice averaged 21 inches in thickness, although areas of hummocked ice up to four feet thick were encountered resting on the river bottom.

An interesting observation of the ice during this icebreaking operation was its change in mechanical properties with changing temperatures. The ice became much more flexible as the temperature rose during the day and this increased flexibility had a measurable impact on its resistance to the ice-breaking. The cooler, more brittle ice was easier to break than the flexible ice (Reference 9), which would often heave up and down a foot or more and remain intact as the wave generated by the LACV-30 passed over it.

6.0 RESULTS

6.1 General

The results presented in this section fall into two categories. The first presents results obtained in accordance with the objectives detailed in the Test Plan. The second covers results of additional tests and specific situations encountered. The latter category covers such subjects as ice classification, ice management, the LACV-30 icebreaking effort for flood control on the Kankakee River, and a summary of the noise level readings for both the LACV-30 and the SUMAC/ACV. Supportive data in the form of summary data sheets can be found in Section 5.3, ice classification sheets in Section 6.7, SUMAC hull survey data in Appendix B and segment failure history in Appendix C.

6.2 Facilitation of Marine Commerce

Results relative to the objective of facilitating marine commerce were obtained by conducting tests in which the SUMAC/ACV escorted towboats and their barges. These tests confirmed the utility of icebreaking operations to commercial towboats. Without exception, towboat operators polled reported that, when the SUMAC/ACV cleared the channel directly ahead of them, their vessels required less horsepower to maintain a given speed and achieved greater speeds at constant horsepower.

No tests were run to determine the ability of the SUMAC alone to facilitate river commerce by escorting towboats. In a number of instances the SUMAC alone attempted to clear the channel but failed to make substantial headway through heavy ice. It appeared realistic to conclude that the SUMAC with its displacement hull, ice deflectors and agitating propwash did contribute materially to successfully clear the channel; however, without the action of

the ACV bow, the SUMAC alone was unable to successfully break the ice in the harsh environment encountered.

During the project, team members developed an understanding of the numerous problems involved in the interaction of towboats and ice and gained insight into possible strategies for alleviating the causes of these difficulties. It was determined, for example, that the ease with which commercial traffic could transit was proportional to the width of the broken channel. On straight passages in the river, commerce was facilitated when a channel wide enough to permit two-way passage was maintained and towboats could further maneuver in an unhampered manner in order to drop off their barges. The widening of the channel at turns reduced transit time by eliminating the need for longer tows to back and fill several times to make the turn.

Because the LACV-30 had no channel clearing capability, it was not as effective as the SUMAC/ACV combination in facilitating towboat traffic through heavily packed brash ice which frequently clogged. In several operational exercises, the LACV-30 assisted barges and boats which were stuck in the ice by breaking ice in the shallow areas close to the stranded vessel, thus relieving the ice pressure and permitting the craft to extricate itself.

Although both ACV's were able to break ice, this was not a sufficient answer to facilitating towboat traffic. Ice clearance was, in fact, as great a problem as icebreaking. The current of the Illinois River was not strong enough to initiate the flow of broken ice downstream; consequently, the ice simply closed back into the channel and refroze. Icebreaking, therefore, is simply one part of a package which must be implemented

in order to facilitate river traffic. This is discussed in Section 6.10, <u>Ice</u>

Management.

6.3 Operational Procedures

The Test Plan called for testing and documenting the operating procedures and maneuverability capabilities for both ACV's. The SUMAC, because of its draft, was restricted to operations only in the channel area of the river where the water depth was sufficient to allow its passage. The LACV-30 craft, however, was able to operate freely above ice and water so its operations were not limited by either the width or depth of the channel. Following is a synopsis of how each craft handled in open water and in ice.

Turning Maneuver

The SUMAC/ACV combination was 169 feet long* and thus required a very large turning radius. The channel width did not permit direct 180° turning but required many direction reversal maneuvers. In open water, turning was usually accomplished quickly and with relative ease. In ice-clogged channel areas, however, turning maneuvers became very difficult. The SUMAC/ACV could turn only as far as broken ice along the side of the SUMAC's stern was loose. Once the ice piled up, the stern could no longer slip sideways and a direction reversal was necessary so that the SUMAC's propwash could loosen and clear an area in the ice into which the stern could slip. With repeated reversal efforts, ice would be moved from one side of the SUMAC to the other and the turn could be effected. However, a 180° turn in ice could take anywhere from half an hour

^{*} The SUMAC was 114 feet long and the RIVER GUARDIAN was 45 feet by 55 feet. During all operations after January 4, 1978, the RIVER GUARDIAN was pushed in its long direction, as shown in Figure 3-1. Thus, the total length of the SUMAC/ACV was 169 feet.

to two hours depending on ice severity.

The LACV-30's ability to turn in a very small radius meant it encountered few difficulties in maneuvering. Neither craft had problems negotiating bends in the river, although the LACV-30 could take the shortest path through a bend while the SUMAC/ACV was restricted to the channel.

Stopping Maneuver

The normal stopping routine of the SUMAC was not noticeably altered by the presence of the ACV bow. This was true even in the one emergency which occurred. The SUMAC/ACV combination was proceeding through refrozen brash ice at about four knots when the ACV's number 2 engine failed and its cushion deflated within six seconds. The SUMAC reversed engines and was able to come to a complete stop in 15 seconds. Despite ice conditions, no damage was incurred to the ACV skirt system.

Stopping maneuvers for the LACV-30 depended upon the ice environment in which it operated. On broken ice or slush, the procedure was to reverse the propeller pitch. On smooth sheetice, pitch reversal was required well in advance of the scheduled stopping point. Additional braking could be obtained by slipping sideways and lowering hover height. If the craft was traveling at high speed, the only effective stopping maneuver entailed the execution of a pirouette. In open water, the stopping procedure is similar to that employed in broken ice; however, the craft can be stopped more quickly if necessary by dropping the cushion pressure and putting both engines on idle. As with the SUMAC/ACV, only one emergency was encountered. In this case, immediate engine shutdown at 35 mph resulted in the LACV-30 coming to a controlled stop within 225 feet.

Passing Maneuver

River operations frequently required that the ACV icebreakers pass commercial towboats and their barges. The SUMAC/ACV combination experienced little difficulty in passing, provided such operations were performed only in the channel where water depth permitted their execution. In open water or moderate ice conditions, the channel width generally varied from 200 to 300 feet and passing posed no problems. In heavier ice conditions such as those found in Peoria Lake, however, passing became a difficult and timeconsuming maneuver with ice, rather than water depth, becoming the limiting factor. As explained in Section 5.5, ice would constrict the channel down to an effective width of about 60 feet. The ice on the outside of this channel became hardpacked and deep and significantly reduced the SUMAC's maneuverability. As the SUMAC/ACV pulled out of the operating channel into the hardpacked ice on the edge, the ice deflectors on its hull acted like a snow plow and began to push the ice forward with the SUMAC until it was no longer possible to push the ice mass any further. At this point, the SUMAC had to back down to clear the ice from under it before further progress could be made. Under these conditions, a passing maneuver could take more than 30 minutes.

Two other interesting observations were made during passing operations with the SUMAC/ACV. First, in heavy ice the SUMAC/ACV was forced to pass rather close to commercial towboats due to the ice-constricted channel. The powerful propellers of commercial towboats sucked in so much surrounding water that on several occasions the force drew the lighter SUMAC/ACV towards the towboat. The SUMAC/ACV then had to perform special maneuvers to avoid

collision. Another potential danger was ice damage to either the vessels or the barges as large chunks of ice were crushed between their hulls. Second, when the RIVER GUARDIAN was operating with a portion of its anti-spray skirt missing, water and ice projectiles were sprayed out 10 to 20 feet from the craft. During passing operations some spray would hit the towboat deck, creating a source of potential injury to towboat crew members on the deck.

Towboats were routinely passed by the LACV-30 in less than one minute. Generally, clearance was no problem since the LACV-30 could travel over shallow areas of the river away from the main operating channel. The only factors that might bear consideration when operating the LACV-30 close to normal river traffic would be ice projectiles from the cushion spray, possible propeller blast, and noise.

Operating Speeds

During operations in heavy brash ice it was found that the RIVER GUARDIAN operated best at speeds of approximately 3.5 to 5 mph, with cushion pressure at 1 psi and a bow-up attitude of between 1 and 1.5 degrees. When operating in heavy brash ice at lower speeds, the flexible skirt system tended to tuck under, causing the bow to dip and rendering the skirt segments susceptible to ice damage. At higher speeds this tucking action did not occur, and the bow skirt segments rode up easily over the ice. The ice modifications on the SUMAC, however, proved to be a hindrance in heavy ice. Its ice deflectors, with their snowplow action, slowed progress and made it extremely difficult to maintain optimum speed. As a result ice accumulated under the ACV's stern, which caused its bow to dip, and necessitated cessation of operations while the SUMAC backed down to clear the ice. In open water the craft could move

easily at 7 to 8 mph and was capable of 4.5 to 5.5 mph in a moderately icy channel.

The LACV-30 operated at a constant cushion pressure of 0.5 psi but its speeds varied from 10 to 35 mph. As will be explained in Section 6.8, the LACV-30's icebreaking capability at high speed (>10 mph) depended entirely on the size of wave it generated and, therefore, wave amplitude was correlated both to speed and to water depth. For open water transits on the river a speed of 35 mph was used in order to minimize the amplitude of the trailing wave. In broken ice and heavy brash, maneuvers were carried out at about 22 mph because it was found that higher operating speeds resulted in accelerated skirt wear and damage. A speed of 12 mph usually corresponded to hump speed (see Section 6.8) and was used during icebreaking operations.

For data gathering in ice conditions, project engineers found the LACV-30 invaluable. Not only did it have high speed maneuverability but it could transit over ice in shallow water regions at speeds of 35 to 50 mph. Without this capability in Peoria Lake, development of ice profile surveys (Section 5.5) would have been difficult if not impossible.

From experience gained during the test and evaluation program, ice-breaking operational procedures were identified that could be generalized for either self-propelled or nonself-propelled ACV's in a river icebreaking role. Since these operational procedures are based on observations and measurements taken during a single testing period, they should be considered as guidelines and not procedures to which either type of craft should rigidly adhere.

Table 6-1 presents a summary of operational procedures for a nonselfpropelled ACV. It was prepared in order to correlate the testing maneuver

ICE ENV IRONHENT HANEUVER	THICK UNIFORM PLATE ICE	HEAVY BRASH ICE
Channel Clearance	Speed should be maintained (4 to 5 mph) to keep the momentum up and to sweep the channel as wide as possible. ACV should be bow up I to 1.5 degrees. Ballasted only as much to equal cushion pressure and ice thickness plus a pressure margin.	Speed should be maintained (3 to 4 mph) as long as cushion area does not become clogged. If clogged, push boat must back down to clear cushion. Bow up 1. to 1.5 degrees. Increase fan flow rate to reduce possibility of tuck-under of skirt.
Enlargement of Turns in Channel	Break out channel width on outside of turns in channel to allow the towboats to swing stern around to expedite turn,	By starting on the largest radius of the turn make an icebreaking run to widen the turn, each one closer in to the inside of the turn. This is to move the heavy brash away from the outside of the turn.
Torboat Assistance	Use channel clearance description. Stopping to allow towboat to catch up if channel closes up too quickly.	Speed of icebreaking should be geared to speed of the toutoat which is being assisted. Because heavy brash closes in much more quickly than plate ice, the icebreaker must operate more closely to touboat.
Passing	When passing a towboat or barge speed should be kept up to maintain controlla- bility.	To pass in a narrow channel one of the two passing vessels may have to stop to allow the other to move around and into the channel ahead or behind passed boat. There is a tendency in brash to be "sucked" into a boat or barge being passed if it is in the channel and the passing boat must go outside.
beaching.	Make a pass along beach to clear ice from beach area. Get a running start to push the ACV up on the beach as far as possible. Stand by to drop spud to reduce slide back.	Same as uniform plate ice.
Turn Around 1800	Wide radius turns should be made with caution because of potential hazard to rudders. Turn in narrow channel should be made by alternate ahead and back movements to move the ice away from the stern in order to make the turn.	A wide radius turn may be made if channel width is sufficient (300 ft.). Full tudder, inside engine full asteam, outboard engine full ahead. If channel width will not permit a full turn, backing will be necessary. Care should be taken to protect turning rudder by keeping them aligned with boat. Flanking rudder may be used for turning.
Backing bown	Turning rudder wast be straight with track. Use flanking rudder if a turn is desired.	Turning rudder must be straight with track. Use flanking rudder if a turn is desired.

Table 6-1 Nonself-Propelled ACV Icebreaking Procedures

with the specific environmental condition in which it was performed. In addition, only those conditions which are unique to the ACV in a river ice environment or which might prove hazardous to its operational ability are cited.

Table 6-2 describes the operational procedures for icebreaking with a self-propelled ACV and, again, is limited to those procedures which are unique to the river ice environment. It should be noted that, since a self-propelled ACV such as the LACV-30 operates in a wave generation mode, it is necessary that the pilot of the vessel be kept constantly informed of the size of the wave he is generating. At present, there is no automated way of monitoring wave size and the pilot must depend on information relayed to him by an onboard observer. Since measurements were taken from the cabin while the craft was underway and visibility from the cabin was frequently hampered by spray on the windows, collected data were often in the form of educated guesses rather than precise statistical records.

6.4 Cost Effectiveness

Originally, the Test Plan called for the evaluation of the cost effectiveness of ACV icebreakers in facilitating river commerce. The most direct means of measuring icebreaking effectiveness was to monitor river traffic through areas directly involved in the ACV icebreaking project. The nature of the project, however, was such that it was impossible to maintain an intensive commerce monitoring effort. In addition, it was realized that the direct assignment of cost values to certain aspects of river commerce would be difficult. Therefore, the requirements of this objective were modified to provide a cost analysis which would include development of a general cost model limited to ACV

MANEUVER	USE/ICE CONDITION
High Speed Low Amplitude Wave	Good for relatively thin ice, 6 to 8 inch wave propagates out bank to bank.
High Speed, Quick Slow Down to Hump Speed	Used on heavy fast ice (>18") to maximize wave for short duration.
Hump Speed	Carries maximum wave. Good for normal icebreaking rate up to 20 mph where ice pressure may be relieved by open water. Also used to reduce size of ice plates once broken out of large ice field.
Hump Speed, Side Slip	Used to carry maximum wave into thick ice and amplify wave at edge of ice field for short distancefrom open water.
Edge Work	Using hump speed technique work edges of river where surface runoff water has weakened ice provided water depth is adequate.
Figure "8" or Crescent Passes	Used on edge of ice from open water where maneuvering room is available.
Criss-Cross	Used for hole enlargement when start- ing from a solid cover ice field repeated criss-cross is necessary to break up ice chunks as small as poss- ible to reduce wave damping.

Table 6-2 Self-Propelled ACV Icebreaking Procedures

icebreaking and an assessment of project costs directly related to ACV icebreaking efforts.

Results of efforts addressing the modified requirements of this objective provided a preliminary cost model specifically oriented toward ACV icebreaking operations. The model consists basically of three components combined linearly to form a total cost by the formula:

Total Cost = Initial Cost + Annual Cost + Operating Cost

Each component can further be defined by dollars, dollars per year and
dollars per hour, respectively.

Initial costs consist of those one-time expenditures made at the beginning of any ACV icebreaking project. These would include the cost of the vehicle, plus facilities, support and induced costs and can be broken down as shown in Table 6-3:

Vehicle Cost	Facilities Cost	Support Cost	Induced Cost
ACV Major spare parts	Land Ramp Building(s)	Major equipment Auxiliary equip- ment Personnel and Management	Ice Management Spillways Structural Pro- tection

Table 6-3 Categories of Initial Costs

As this table illustrates, it is unrealistic to regard ACV icebreaking costs as consisting only of the vehicle and initial spare parts costs. Facilities costs must include those for a land area where beaching and repair tasks are conducted. The site should offer a suitable ramp to allow the ACV to enter and exit the river and buildings for equipment storage and craft shelter.

Personnel costs for vehicle operation and maintenance can be established from standard Coast Guard billet models. It will be more difficult, however, to assess the cost of project startup in terms of management. In order to have a successful operation, extensive planning must be done to assure efficient coordination of operational and maintenance efforts. An inventory system for ordering and storing spare parts will also be required.

In addition to personnel and management costs, other categories of support costs are identified in Table 6-4:

Major Equipment	Auxiliary Equipment	Management
Mobile crane Hot bonding equipment Generator Welder Air compressor Transportation vehicle - truck	Tools Spare parts Winches Ballast Rope and line	Start-up Coordination Inventory Storage Office equipment Bookkeeping

Table 6-4 Categories of Support Costs

Results of current icebreaking operations suggest that certain costs which are induced by such an operation bear consideration. Induced costs would result as a direct product of icebreaking efforts. For example, control of ice interference with lock and dam operations and ice damage to river structures will induce added expenses and will require some type of ice management program. A proposed ice management program is discussed in more detail in Section 6-11.

While it is possible to predict an average annual cost for ACV icebreaking efforts, the rate of operating costs depends to a great degree on craft integrity and the ice environment. In mathematical form, the operating costs component of the model may be expressed as:

Cost/Hour = Fuel Gallons x Cost + Segment Failure x Cost + Maintenance Costs
Hour Gallon Hour Segment Total Hours of
Operation

In developing the cost model, the life of the equipment was assumed to be 10 years and operating costs were reduced to dollars per hour. Since the project was primarily research-oriented, it was not possible to assign dollar values to all the components involved in its completion. No permanent facilities were construced, the beaching ramp was merely some bulldozed dirt territory, and much of the equipment was supplied by the U.S. Army Corps of Engineers. Furthermore, both craft were available on some form of leasing arrangement. Given these limitations, the following is a proposed total cost model:

Total Cost (\$/yr) = Initial Cost + Annual Cost + Operating Cost where

Initial Cost = Vehicle Cost + Facilities Cost + Support Cost + Induced Cost
Annual Cost = Major Annual Maintenance Expense

Operating Cost = Fuel Cost + Skirt Replacement Cost + Repair Cost

Tables 6-5 and 6-6 present worksheets for a suggested model which may be applicable to future ACV icebreaking studies. As can be seen from the categories presented in the tables, itemized costs can be put in the model to provide total cost as follows:

Total Cost (\$/yr) = Initial x 1 + Annual + Operating x m hrs Cost (\$) TO yrs Cost Cost (\$/hr) yr

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A.		Dollars
	ACV	\$2,500,000.00
	Spares (Estimated 10% Craft Cost)	250,000.00
	Total	52,750,000.00
8.	Facilities Cost	Dollars
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	Ramp	
	Building(s)	
	Total	
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Table 6-6 Cost Summary Worksheet for A Self-Propelled ACV

A precise cost determination could not be made for the test and evaluation program. This was mainly due to the fact that many facilities, equipment and support items were contributed by either the Coast Guard or the U.S. Army without direct costs being assigned to the project.

6.5 Facilities/Personnel Requirements

An in-depth study of personnel skill requirements versus Coast Guard training requirements was beyond the scope of this evaluation. Coast Guard personnel from the SUMAC, the Second Coast Guard District and the R&D Center, familiar with and experienced with Coast Guard personnel skill levels in comparable jobs onboard diesel and gas turbine driven cutters, observed the ACV operations at close hand. In their judgment, both the nonself-propelled and self-propelled ACV's can be maintained and supported by Coast Guard personnel after special training specifically related to the two vastly different craft types.

Maintenance and support facilities would appear to be more significant obstacles to successful operations because of the severity of the river ice environment. The Coast Guard must therefore be prepared to provide special facilities and support equipment tailored to maintain and repair each type of ACV. Since both must be maintained and repaired in unimproved outdoor locations, it is important that adequate portable support facilities be provided.

There were, for example, numerous segment failures on the RIVER GUARDIAN during the test program. After operations in which damage was incurred to the skirt system, the craft had to be beached at the repair site near Peoria

Lock and Dam. This site was simply a bulldozed dirt ramp with a dirt area at the top where both ACV's could be parked for repair. The beaching operation for the RIVER GUARDIAN was conducted by using a Caterpillar tractor and a winch truck to pull the craft up the beach incline and to maneuver it on land so that maintenance personnel could have access to the area under the craft which required repair. This work was often performed in freezing weather on poor terrain with only makeshift tents and portable heaters to provide some element of comfort. As a result of these beaching and repair operations, it became evident that more emphasis should be placed on considering the supportability and maintainability of the ACV in the field.

There are considerable differences in support requirements for nonself-propelled and self-propelled ACV's. Clearly, since the nonself-propelled craft works in tandem with a sizeable pusher vessel, some repair equipment can be carried aboard the latter. Emphasis should be placed, therefore, on providing maintenance support for the ACV while it is actually away from port.

Table 6-7 is a partial list of suggested facilities and equipment necessary for the supportability of a nonself-propelled ACV. It is divided into items that would be located at ACV beach facilities and portable items that could be carried on board the pusher vessel.

The LACV-30 posed two problems which were not present in the nonself-propelled ACV. First, it can run at high speeds and thus range a considerable distance away from any land-based maintenance and repair facilities. Secondly, since it is self-contained it does not have the physical capacity to hold large amounts of equipment or to carry extra crew specifically assigned maintenance and repair tasks. Experience gained in the early part of the

FIXED BASE ITEMS

Repair and servicing ramp area, hard surface with shallow graded ramp from water's edge
Capstan winch or winch truck* for beaching
Tugger winches for stability control during ACV movement on land
Storage and repair building(s)
Crane truck* or equivalent to handle lifting of segment frames, etc.
Electric welder and related barge repair items
Sewing machinery and hot bonding equipment for repair of rubber skirts

PORTABLE ITEMS

Heaters with fans and tent covering for worker protection from weather during repairs
Diesel engine maintenance tools
Electric/pneumatic hand tools, e.g., impact wrench, riveting gun, drill Portable segment frame handler
Assorted tools for pulling and repairing segments
Deck ice removal gear, ice chippers, rubber hammers, etc.
Heavy duty industrial stapler and segment repair kit

Table 6-7 Equipment for Nonself-Propelled ACV

^{*} Crane and winch trucks can also travel to remote beaching sites when repairs away from base are required.

test program provided a basis for the choice of equipment which was carried aboard the LACV-30 as it traveled from Peoria to the Kankakee River for flood control icebreaking operations. Table 6-8 presents a list of both fixed base and portable items required to maintain and support a self-propelled ACV.

As has been stated, operation of either type of ACV icebreaker is well within the capability of existing Coast Guard personnel and technological levels. Coast Guard crews, with their basic knowledge of vessel operations, maintenance, and repair, would be adequately qualified to handle the bargetype ACV. The only requirement would be to gain some elementary background on air cushion dynamics and the ACV skirt system characteristics. Most training could be completed on the job and should involve both operators and maintenance personnel, since it is important that each component of the crew learns to recognize the first indications of any malfunction in the craft. With this capability in hand, personnel could probably perform more repair tasks while the vessel was operating and avoid time-consuming and complicated beaching operations.

The LACV-30, on the other hand, is a complex piece of machinery which requires a trained crew. This craft requires a trained operating crew and a maintenance crew capable of working on turbine engines and flight systems such as propellers and aircraft navigational controls, in addition to less technical tasks such as skirt repair and replacement. The Coast Guard would have to initiate special training programs to give personnel direct experience in operating and maintaining an ACV of the LACV-30 type.

FIXED BASE ITEMS

Building and Yards

Building and yard facilities should be designed to allow the craft to fly from the river to the service area. The ramp must be graded to assist in maintaining and guiding the craft in a prescribed track. The low speed maneuverability of these vehicles is marginal and they tend to seek the path of least resistance. Any grade in the transiting surface will cause the craft to drift.

Machinery

It may be necessary to lift at least 2/3 of the weight of the craft and to place blocks under the landing pads so access may be gained to the bottom. A crane of the required capacity should be available. It will be necessary to move the craft in areas where good surfaces are not available. This requires a tow truck to pull the craft around. In addition, another vehicle is needed to stabilize the craft from the stern. They should be of such weight and bollard capacity to handle the craft in the expected terrain.

PORTABLE ITEMS

Skirt repair equipment, i.e., riveting tools, welder, etc.
Air compressor
Brush clearing equipment (chain saws, axe, etc.)
Engine repair tools
Portable heaters and covering to protect workers during repair periods
Fuel supply

Table 6-8 Self-Propelled ACV Facilities and Equipment

6.6 Design Criteria

This section discusses design problems which became apparent during the testing of both the self-propelled and nonself-propelled ACV's.

Skirt System Material

One of the major design considerations to come out of the icebreaking test and evaulation program was the effect of the harsh ice environment on the skirt system material. The skirt material employed on the RIVER GUARDIAN weighed 60 oz/sq yd. It was made of rubber reinforced with a single weave nylon fabric. During the project this material proved to be inadequate for the severity of the ice encountered. The LACV-30 employed materials of several different weights. These materials consisted of neoprene reinforced with a two-ply weave of nylon fabric and did not have the high failure rate of the naterial used on the RIVER GUARDIAN.

Tensile strength and tearing tests were conducted at Virginia Polytechnic Institute (VPI) on sample skirt material from both craft and the two-ply material proved to be of greater strength. It should be pointed out, however, that there is no basis for any direct operational comparison between the materials used on either ACV. The RIVER GUARDIAN had a straight finger-type skirt design and operated with much higher cushion pressure than the LACV-30, which had a bag and finger-type skirt formation and operated at much higher speeds.

To extend the life of the skirt segments on the RIVER GUARDIAN, liners were installed inside the bow segments. The bow area had experienced the

highest rate of segment failure as evidenced in the damage diagram shown in Appendix C. The liners, which were made of polyvinylchloride (PVC) bonded over a nylon fabric, contributed significantly to the improved operational life of the RIVER GUARDIAN's skirt system. In addition, a bumper system of cut tires was added over the bow skirt section to reduce the impact of ice on the skirt. It was not readily apparent to what extent this bumper system improved skirt segment life but some type of bumper system might be considered in future nonself-propelled ACV designs.

Spray Suppression System

During icebreaking operations ice accumulated on the decks of both the RIVER GUARDIAN and the LACV-30. The amount of ice that accumulated was directly related to the amount of spray generated by the craft.

The RIVER GUARDIAN had a spray suppression skirt configuration which contributed significantly to minimizing the amount of spray generated by the air cushion. Such a system would have been desirable on the LACV-30 and should be a design consideration in future ACV designs.

Ice accumulation proved to be a problem which often interfered with operations. Ice buildup on the forward deck of the RIVER GUARDIAN contributed on occasion to its bow down attitude. Also, ice accumulated on the fan intake structure of the RIVER GUARDIAN and, at one point, effectively reduced the hover height by constricting the air flow.

Ice accumulation due to spray directly affected the operations of the LACV-30 by impairing visibility. Because the LACV-30 used the wave generation

mode of icebreaking, it was important to monitor closely the speed of the craft and relate this to the amplitude of the wave it was generating. On some occasions, personnel on the LACV-30 could not clearly see the characteristics of the wave due to marginal visibility which resulted from spray freezing to the windows of the craft. In addition, only the forward windows of the craft were equipped with windshield wipers.

Other Observations

Many other items relating to ACV icebreaker design criteria were developed during the project. Table 6-9 presents suggested design criteria items for a nonself-propelled ACV and Table 6-10 presents design criteria for a self-propelled ACV.

- Dimensions of craft should be compatible with barge size for optimum channel clearance, e.g., 45 ft. by 55 ft. or greater
- o Minimize above deck clutter of equipment and structure to reduce problems of deck ice accumulation and to reduce sail area
- o Spray suppression skirt should be included in the design
- o Cleats, chocks, bits and pad-eyes should be designed for beaching operations and for facing up to towboat
- o Push pads and restraining rigging must be designed to allow for movement of the ACV during hover height changes, but should restrain ACV so as not to allow a bow down attitude thus preventing plow-in
- o Skirt material should be 90 oz/sq yd, or heavier, advisably of a double weave construction with high tear resistance
- o Skirt segment design should include rip-stop fabrication. The back of each segment should be enclosed to minimize ice scooping and to maintain pressure in the segment greater than the cushion.
- o Means for varying cushion pressure (ballasting) should be provided. Ideally cushion pressure should be adjustable over a range of 0.8 psi to 1.5 psi.

Table 6-9 Nonself-Propelled Craft Design Criteria Items

- o External wipers should be provided to help keep all windows clear of spray and ice
- A spray suppression system should be installed to minimize spray
- Deck clutter should be kept to a minimum in order to reduce the area where ice can accumulate
- A deck winch system should be included to assist in pulling the craft free during possible ice groundings
- Crew comfort during long missions should be considered and areas for food storage, etc., provided
- O A variable ballast system should be installed, which would permit cushion pressure and center of gravity location to vary. The suggested cushion pressure range is 0.3 to 1.0 psi.

Table 6-10 Self-Propelled Craft Design Criteria Items

Items from Table 6-9 which might require further explanation include:

- 1. The spray skirt design on the RIVER GUARDIAN provided no rip stops and, consequently, when a puncture or tear in the material began, it quickly grew and extended from the top to the bottom of the segment. Future designs should incorporate some rip stop seams to localize rips and prevent their spread, thus enabling craft to maintain hover height.
- 2. Each segment of the RIVER GUARDIAN was enclosed in back to prevent it from scooping up broken ice as the craft moved over the ice field (See Figure C-1, Appendix C). Unfortunately, however, these backs were usually ripped off the segments by the harsh ice. Further study is needed to fully understand the effects of this design feature on skirt life and cushion pressure.
- 3. Minor effects on the steering of the SUMAC were often noted when cross channel winds hit the ACV bow. It might be desirable, therefore, to reduce the sail area on a barge-type ACV by placing the fuel tanks and engines below

deck, if possible. It should also be noted that by reducing the number of items on deck (deck clutter), the number of points where ice can accumulate is effectively reduced.

An item from Table 6-10 which might require further explanation follows: The fact that ice often protruded four to five feet above the water surface can be seen in the ice channel profile (Figure 5-2, Section 5.5). During one operation the LACV-30 was approaching such an ice mass and reduced speed, hoping to avoid it. Unfortunately, the craft did not stop in time and came to rest on the ice mound. There it became high centered and was effectively grounded. Efforts to free the craft using on-board deck winches were ineffective, and the help of a commercial towboat in the area had to be enlisted. Future ACV designs should include sufficient deck equipment to free the craft in similar situations.

A more complete treatment of design criteria is presented in Reference 9. The information is divided into five areas: hull, power, life arrangements, auxiliaries and supportability. It is not intended to be a complete set of specifications for the design and construction of an ACV icebreaker, but a presentation of design criteria items that are considered as necessary or desirable in ACV icebreaking operations.

6.7 A Method for Characterizing River/Lake Ice (Relative Resistance Factor)

During the test and evaluation program there was a need to compare results from various locations and time frames of test runs. In addition, ice environments needed to be compared to determine the effectiveness of icebreaking under varying craft configurations. As a result of this, a standardization of river ice environment was achieved by developing a

Relative Strength Factor (RSF) method of characterizing the ice. This methodology employed measurement of specified ice parameters such as thickness, plate size, plate spacing, and knitting thickness. It also considered the classification of the total ice field, e.g., melting brash, uniform cold ice and similar conditions. The resulting procedure produced a number which described the unique river ice condition during a particular test situation. This permitted comparison of test data and icebreaking effectiveness over a span of time and locations. Equal RSF values indicate ice conditions which present similar resistance to breaking and/or resistance to channel clearance.

Table 6-11, Ice Parameter Values and Unit Strength Factors, is the basic form used in developing the relative strength values for each test. This form and procedure were used by various test engineers during the test program. Results were compared between recorders and a weighted averaging resulted in the final RSF value. Tables 6-12a, b, c, and d contain the ice classification forms used to record the ice parameters, field type, comments, and calculated resultant RSF.

6.8 Icebreaking Mechanisms

In general, ACV's break ice in two different modes depending on their speed and cushion pressure and on the type and thickness of the ice encountered. The two modes are wave generation and water depression. Since the nonself-propelled ACV operates mainly at low speeds with a high cushion pressure, it breaks ice in a water depression mode. The self-propelled ACV usually operates at high speeds with low cushion pressure, and thus breaks ice in a wave generation mode.

thickness of knitting holding the ice mass together. Select the unit strength factor (U.F.) for the existing ice. Next by observation determine ice field type and select U.F. for that type. Add By measurement determine ice parameters, thickness of plate, size of plate, plate spacing, and up all U.F.'s. The resulting value represents the relative strength of that ice field and is a measure of resistance of the ice to breaking and displacement. INSTRUCTIONS:

ICE PARAMETERS		U.F.*		U.F.		U.F.		U.F.	The state of the s	U.F.	<u>¥</u>	U.F.	U.F.
Thickness (Avg.) of Plate	Inches (1-4)		Inches (5-8)	2	Inches (9-12)	8	Inches (13-15)	4	Inches (16-19)	5	Inches (20-25)	. 9	Over Use 7
Plate or Grain Size	Feet (1-4)	-	Feet (5-8)	2	Feet (9-12)	3	Feet (13-15)	4	Feet (16-19)	2	Feet 5 (20-25)	9	Over Use 7
Plate Spacing	Inches (1-4)	S	Inches (5-8)	4	Inches (9-12)	3	Inches (13-15)	2	Inches (16-19)	-	Inches (20-up)	0	
Knitting Thickness	Inches (1-2)	1	Inches (3-5)	2	Inches (6-8)	3	Inches (9-11)		Inches (12-up)	2	-		
* II.F. = Unit Strength Factor	trength F	actor										S. Jonathama	

Uniform Thickness Unbroken - Cold
Refrozen Brash/Plate - Cold
Uniform Thickness Unbroken - Melting
Refrozen Brash/Plate - Melting
3
Unconsolidated Brash

Table 6-11 Ice Parameter Values and Unit Strength Factors

Relative Strength Factor (RSF)		DESCRIPTION AND COMMENTS	11 *Brash field unconsolidated - reconstructed data	*Brash field unconsolidated - reconstructed data*	Brash consolidated ice up to 14 inch average 6 - 8 inches, estimated ice data	13 "Uniform unbroken ice - reconstructed data"	Uniform unbroken ige in Peoria Lake - reconstructed data	Brash consolidated some imbedded plates - reconstructed data	New ice very closed packed field	Ice measured 29 cm thick snow cover 2 inches	Brash ice next to already broken channel at beginning of test	We came into a ridge of ice piled up by the barges	This ridge was estimated to be 3 to 4 feet thick.	Very thick slush and plate ice.	Show cover 3 inches, ice very thick.	Raffed up, 1 foot thick.
	LAE	RELAT	==	* 6	12 +	13,	16	12	16	18	11	14	13	14	16	10
	BEASH SOLI-	DATED	-	-												
34		BRASH			3			3								
FIELD TYPE		UNIFO					2									
FIE		COLD														
	ICE BW	COLD				•			-	-	2		3	3	~	2
	ness(in.)	THICK	%	%	1/	7/	3/	2/	9	3/	7-	7-	7	7-	7	7
TERS	NG(IN.)	PLATE	3/	2/2	1	7	0/	9/	7	/	3/5	-/5	7/5	7	2/3	= 7
PARAMETERS		CEATE PLATE	7	5/2	1	-/-	2	1/-	-/-	7/2	7	2/2	1/	9/	3/-	7
088	NESS ATE(IN.)	THICK	9/	7	0/	7	2/	°/	9/2	3/2	7	2/	7	=/-	=/	2
		N.N	-	-	-	-	345	3-8	-	~	-	1. E	7	6	162	-
		T.C.	600	700	900	900	800	600	010	010	110	110	1110	110	012	017
		DATE	1977		1978	1/11	1/16	1/13	1/31	1/51	1978		1/2)	1/23	1/24	1/24

*Relative strength factors reconstructed from test records not determined directly using RSF in characterization technique.

Table 6-12a Relative Strength Factors

(424) source discounts authorized		DESCRIPTION AND COMMENTS	Raffed ice, 2 feet thick, heavy.	Raffed ice up to 10 feet thick in some places.	Ice was unconsolidated brash ice, provided little resistance, .9 cover.	Ice brash about 3 feet thick along side of SUMAC 2 fc. in front of ACV, resistance relatively high.	Brash ice 3 to 4 ft. ahead, 5 ft. along side, plate imbedded in brash.	Brash unconsolidated 2 ft. thick imbedded chunks 10 in. thick 4 to 8 ft. across	Brash unconsolidated 3 ft. thick imbedded chunks 8 to 10 in. thick, 8 ft. across	Brash ice beginning to consolidate, photo 2 to 3 feet thick.	Brash ice with balls of ice 5 feet thick	Brash ice with ice balls 10 ft. thick, slush	Plates of ice broken into 3 ft chunks	Consolidated brash and plates	Consolidated brash of uniform thickness & 4 inches
3				_		Ice brash a	_				Brash ice w	-	Plates of 1	Consol idate	Consolidate
		RELAT:	14	15	2	4	9	9	9	=	12	-23	2	17	2
		писои			-	-	-	-	-	-					
IPE	NC ON	REVER									3	6			
FIELD TYPE	NC SM	NELTI													
PIE		COLD												4	4
	3DI	COLD	3	-									3		
9	(ESS(IN.)		~/_	~/_	7	0/	0/	%	0/	7	~/	~/	7	~/	7
ERS	(IN.)		-/	-/	2/2	6	=/	7/6	2/2	FZ	=	=7	7	5	6
PARAMETERS	TA SEE	PLATE	0/	6/	=/	-/	2/	۵/	67	£/	=7	=7	2/	6/	4
PA	RESS (IN.) STA		07	2/2	6	10-	6	9/	67	27	27	27	15	67	6
9	IESS		1/-	1	/~	/-	1	/~	1/2	1-	1	1	1	/~	1
		1	*	~	-	7	3	1	~	-	4	~	-	2	3
		7.C.	610	017	013	610	013	910	014	910	710	110	910	910	910
		DATE	1978	1/24	2/3	2/3	2/3	2/4	2/4	2/4	7/7	7/2	3/6	9/2	3/6

Table 6-12b Relative Strength Factors

(200) roses Research Forest		DESCRIPTION AND COMMENTS			Uniform thickness with chunks measuring 20 inches average 18 inches	This reading was in the channel and taken on the	l rubbl	Rubbie 20 Inches thick 12 to 14 feet, 90% cover *this should be RSF=20	Lightly consolidated uniform thickness plates	Lightly consolidated uniform thickness plates	Thin consolidated brash	This measurement covers runs 2, 3, and 4	Consolidated brash fairly uniform thickness in channel	Raffed in areas immediately outside. Runs 16 and 17 same	N.A.	Rubble chunks 6 slush balls with plates 4 - 6 in.
	IAE	RELAT STREN	15	20	23	11	=	=	10	10	=	13	13	77	14	₹
	FEASH SOLI-	DATED						-								
1 adi		BRASH									9	3	3	3		
FIELD TYPE		UNIFO		2					2	2					2	
I I		COFD BBV2H	4													
		COLD			3	3	3									86
	NEZZ (IN) INC	KNITT	9/5	2/2	9/1	7-	/~	%	7_	7-	7-	7/2	7/2	7	3/2	7
TERS	(.иг) эи	PLATE SPACE	7	9/1	7	75	7/2	170	=/-	6/3	1/2	1/2	1/2	2/2	1/5	4
PARAMETERS		PLATE WIASO	1/-	9/	2/3	1/-	7-	22	2/-	7/2	1/-	7/-	7-	7/-	2/2	7-
	NESS ATE(IN.)	THICK	9/2	7/4	چ/ء	7-	/-	2/3/	7-	7-	/-	2/2	2/2	2/8	2	2
		2	9		1	3	4	5	-	2	-	2	5	1	1	1
		7.C.	910	910	017	017	017	110	023	023	024	024	024	024	025	028
		DATE	1978	2/6	1/2	1/1	1/1	1/1	2/10	2/10	2/11	11/2	2/11	11/2	2/11	1/21

Table 6-12c Realtive Strength Factors

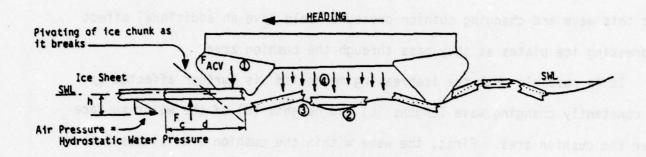
	V.						185									
actor (RSF)	187 187	2 23 3 5 W	ld (location	ne to	95	i sd	97			93						
Relative Strength Pactor (RSF)	ane uda ane	COMMENTS	and rubble file	in a		iok idt	W		35		30		15 15 15			
Relat	948) 1515	DESCRIPTION AND COMMENTS	Refrozen brash and rubble fleld (location Beardstown 97.9 mile)	Refrozen channel	70	, b	a f			270			6V1			
	RID	RELAT	13	15	000											T
	BKYZH															
2		BETTE														Г
FIELD TYPE	NC KW	MELTI														
LE	201	COLD	4	4												
	ICE KH	COLD							361			000				
	NEZZ(IN) INC	THICK	7	7/2	/	1	1	1	1	1	/	1	7	1	1	1
TERS	(·NI) ON		9/4	2/2	1	1	1	1	1	1	/	1	7	1	7	/
PARAMETERS	SIZE (FT	PLATE	1/-	2/2	/	1	1	1	1	1	1	1	/	1	1	1
	NESS VIE(IN.)		3/5	3/5	/	1	/	/	/	1	1	1	1	1	1	1
	ženená.	*	~	6								10	1	é il		
	al Ji	7.C.	028	028	0			0.71		1000	0.0	2 61		0 1		
	277 04	DATE	1978			25			50					ma	Phys	

Table 6-12d Relative Strength Factors

YICHW DERES

While the above mechanisms are accepted as standard, there exist other phenomena in each icebreaking mode that have not been carefully studied, such as those associated with the air pressure field moving across an ice sheet. If one analyzes the plenum characteristics and air dynamics of an ACV cushion relative to the response of the water surface under the pressure field, three factors which influence icebreaking capabilities can be identified. The first is the depression of the water surface under the leading edge of the ice, which leaves the ice block extended as a cantilever beam. The second factor is that a wave is always generated by the advancing pressure field. The third is that cushion pressure fluctuates at a high frequency. These physical phenomena were identified during over 400 hours of testing the SUMAC/ACV and the LACV-30 and were found to be common to both vehicles.

Careful analysis of actual icebreaking mechanisms in the water depressing mode revealed that it is not enough simply to hypothesize that ice, acting as a cantilever, breaks downward due to the burden of its own weight (Reference 11). Actual observations indicated that as the bow skirt section of the ACV moved onto the leading edge of the ice field, the air from the cushion was forced ahead of the craft, displacing the water under the ice by a distance d. The gauge pressure between the ice and the water below it is equal to a hydrostatic head of water h (where $P_g = \rho gh$). The strength of the ice may be exceeded by this applied pressure so that, when it is broken, it actually bends upward rather than falling downward due to its own weight. In other words, the force of upward air pressure combined with the downward pressure exerted by the craft form a bending moment, causing the ice to pivot and break off. Figure 6-1 presents a rudimentary force



- As the ACV moves forward, the bow segments press down on the face of the leading edge of the ice sheet. The water is forced out from under the ice, causing the ice to cantilever. The ice thus fails from the moment created by the ACV bow pressing down and the air pressure pushing up.
- 2. The predominant icebreaking mode of the self-propelled ACV is its ability to generate a large stern wave by using low cushion pressure at or near hump speed. As the wave passes through the cushion, it places hogging and sagging loading on the ice chunks which then fail due to cyclic fatigue.
- 3. As ice chunks collide in the cushion area, they are further broken into much smaller pieces.
- 4. It is possible that the high frequency variations in the cushion pressure are near the natural frequency of the ice. This may cause the ice to vibrate, accentuating the thermal cracks and causing additional failures.

Figure 6-1 Diagram of Icebreaking Mechanism

diagram which illustrates this phenomenon.

In addition to the above icebreaking mechanism, the water depression mode has several other interesting aspects. As the ACV moves forward, the advancing pressure field creates a wave. Its length and height are influenced both by the speed and the cushion pressure of the ACV. It is proposed that this wave and changing cushion pressure field have an additional effect or breaking ice plates as they pass through the cushion area.

It is possible that the icebreaking mechanism is further affected by the constantly changing wave lengths (L) and heights (H) of the water surface under the cushion area. First, the wave within the cushion area places ice plates passing under the craft through cyclic and variable loading patterns, causing them to flex in hogging and sagging actions. Secondly, the waves generate a motion which continually impacts the ice, causing it to break into smaller pieces.

Another phenomenon which might contribute to ice failure is high frequency vibration induced by cushion dynamics. These vibrations may possibly be near the natural frequency of the ice, causing it to vibrate at or near resonance. No data are presently available to verify this hypothesis, but Figure 6-1 presents a suggested conceptual diagram and a summary of a proposed icebreaking mechanism model.

For the wave generation icebreaking mode, the wave generated by the craft is the primary icebreaking mechanism and its character depends upon the craft's speed and cushion pressure, the prevailing ice conditions, and the depth of the water. At low speeds, the bow wave preceding the craft is small and cushion pressure is inadequate to depress the water level and cause the ice to cantilever. As speed increases, however, the craft

actually overtakes the bow wave and eventually rides up over it. "Hump speed" is the term used to describe this condition, which is accompanied by a stern wave of large amplitude, and it is considered the optimum condition for effective icebreaking. It is important to note that if the speed of the craft exceeds hump speed, icebreaking effectiveness is decreased since the vessel actually outruns the wave and causes its amplitude to lessen.

Since wave amplitude is the predominant icebreaking force in the wave generation mode, the wave length (L) and height (H) directly dictate the amount of ice broken. Cyclic loadings from hogging and sagging actions take place in the wave behind the craft, and impact between chunks of ice also adds to the reduction of plate size.

An important parameter identified with both modes of icebreaking is the nature of the ice environment, including the type of ice and the characteristics of the channel. While icebreaking in lakes generally involves uniform ice and unrestricted channel width and depth, the river ice environment presented challenges of a different nature. For example, SUMAC/ACV icebreaking operations on the Illinois River were confined to the channel area where the ice consisted of refrozen brash of porous consistency. In contrast to uniform plate ice which can be cantilevered by air cushion pressure, the porous brash ice absorbed and dissipated air through its cavities. Water forced out of these cavities was evident as spray points visible 50 to 100 ft. away from the ACV. Similar to the cantilever beam effect described above, this air pressure in the cavities helped break ice apart as the ACV rode up over the leading edge of the ice. Wave action

and agitation under the cushion area further helped reduce the size of the ice chunks. The SUMAC's propellers were also responsible for grinding ice while its "V" shaped ice deflectors and powerful propwash helped displace the broken chunks to the sides of the channel. The ice would soon close in behind the SUMAC/ACV combination, however, when the propwash subsided due to the lack of currents strong enough to wash the broken ice downstream.

The barge-type ACV responds in different ways when it encounters ice. This response is dependent on the amount of ballast, trim and cushion pressure which the vessel carries and also upon whether the ice field is of uniform thickness, is consolidated or unconsolidated brash, and to what extent it covers the channel. The nonself-propelled vehicle tends either to ride up on the ice prior to breaking it or to pitch down by the bow and plow in. To prevent unnecessary plow-ins, this type of ACV should be ballasted bow up 1 to 1.5 degrees with cushion pressure maintained near 1 psi. Should all these conditions exist, the ACV generally behaves in the following manner:

Upon contact with a large unbroken ice field of uniform thickness, the bow of the nonself-propelled ACV pitches upward and rides over the ice. Air progresses forward of the skirt system and depresses the water surface under the ice. Portions of the ice field fail and large broken plates tilt upwards, venting the air pressure beneath them. The ACV then pitches back down to equilibrium as the broken ice chunks of approximately 10 to 15 feet across pass into the cushion.

Expectations were that the self-propelled LACV-30 would be able to use its wave generation capability to break ice from riverbank to riverbank.

This. however, did not always happen because of characteristics of the

channel. First, the water depth changed greatly across the 200-300 feet wide channel, from 18 to 30 feet in the main channel to from 0.5 to three feet at the shore. Since wave propagation is dependent on water depth, amplitudes generated in the deep water of the main channel were damped out in the shallow waters along the banks. Second, waves generated in open water or brash ice were damped out by solid plate ice at the edge of the channel. The harsh nature of the brash ice in the operating channel and the lack of current to flush broken ice meant that waves generated by the LACV-30 were not effective in clearing the channel. It broke plate ice on the edge of the channel, but this was of little use to commercial traffic because the water depth there was too shallow for passage. In addition, when the LACV-30 broke ice bordering the channel in Peoria Lake, the chunks sometimes moved into the main channel and created more problems.

Thus, it can be seen that the river ice environment offers several new factors to be considered in the field of ACV icebreaking. The entire field of ACV technology is relatively young and some ideas on icebreaking mechanisms are presented for the first time in this report. Observations made over the period of the study have explained some of the physical responses of the vehicles and the ice as it was broken. Further study of the icebreaking phenomenon under more controlled conditions is now necessary to provide knowledge which could be of fundamental importance in designing more effective ACV icebreaking vehicles.

6.9 <u>Icebreaking Effectiveness</u>

A rational criterion of measuring icebreaking effectiveness is difficult to establish. It would be a sweeping simplification to define

effectiveness of icebreaking as the area of ice broken per hour. First, it is necessary to identify where the ice was broken, since operations in the channel area would obviously have more effect on river commerce than those in shallow waters. Secondly, it is important to consider the quality of the broken ice. Small pieces of broken ice are harmless while large pieces could present a danger for downstream traffic and could even clog locks some distance away. Another factor, first recognized by the Canadians, is the amount of resistance that the ice exerts on the icebreaking craft. Resistance decreases as the size of the broken ice chunks decreases.

The resistance is reflected to a degree by the amount of energy consumed by the icebreaking vessels. In this project, the SUMAC broke ice with and without the ACV on its bow. Data were gathered on the craft in both operating modes and are presented in Table 6-13. The test runs were purposely planned in order to assess performance in RSF ice environments which were essentially the same; hence, the ice resistance factor could be considered as constant.

To illustrate comparative icebreaking performance with and without the ACV bow, consider data items 7 and 8 in Table 6-13. Item 7 records the action of the SUMAC alone operating at an engine rpm reading of 633 from open water into an ice field with an RSF of 15. It failed to make any headway and, in fact, came to a full stop within 1.5 boat lengths. Data item 8 records a run in which the SUMAC with the ACV bow attached encountered an RSF 15 ice field and, with an rpm setting of 759, was able to pass through it at 3.1 mph. Item 9 reflects a continuation of item 7 and shows that the SUMAC alone in the ice field required 1153 rpm with 1955 hp in order to pass through the ice field at approximately 3 mph. Although it succeeded in making headway, it is important to note that its fuel consumption was

DATA ITEM	RPM	н.Р.	FUEL GPH	SPEED MPH	WITHOUT	WITH	R.S.F.	T.C. NO.	FOOT NOTE
1	1066	1671	97.0	6.2	Х	V35. er	12	009	*
2	1105	1607	92.8	4.7	at of the	х	13	024	DREIE
3	897	1062	60.0	6.5	х	Fyed I	10	023	
4	851	585	39.2	3.6	EF sold	х	13	016	90-107
5	1154	1746	111.7	4.2	х		12	009	*
6	1118	1594	94.6	4.7	ande di d	х	14	024	
7	633	316	18.8	N.A.	Х	130929	15	012	**
8	759	500	28.8	3.1	e of ust	х	15	016	upa 0
9	1153	1955	115.5	2.9	х		13	012	***
10	921	857	50.3	3.1	si dowli	х	12	014	el) se

^{*} Estimated RSF

Table 6-13 Comparison of Icebreaking With and Without ACV

^{**} The SUMAC took a running start and hit ice at an estimated 4 mph. The boat stopped dead within 1 1/2 boat lengths.

^{***} This reading was taken after the reading in item 7 and was the power required to make headway in the ice.

115.5 gph. This must be compared with data item 10 in which the SUMAC/ACV combination passed through a similar ice field at 921 rpm and 857 hp with fuel consumption of only 50.3 gph. Overall, the SUMAC consumed 44 percent less fuel when it operated with the ACV bow. It should be noted that this was for the SUMAC alone. It was not possible to directly record the fuel consumption of the RIVER GUARDIAN itself since it varied greatly depending on the ACV engine rpms. Calculations of the RIVER GUARDIAN's fuel consumption, however, revealed an average of 30 gph.

Table 6-14 represents a summary of icebreaking rates under various ice conditions with the rate represented in acres per hour (R_A) where 1 acre equals 43,560 square feet. These data can be analyzed by comparing the rate of icebreaking (R_A) with the ice environment (RSF) and the rate (R_A) vs the power consumed (hp). All data items reflect icebreaking runs with the SUMAC/ACV combination, except items 5, 16 and 18. It can be seen that the SUMAC/ACV was capable of breaking ice at a rate of nine acres per hour in ice with an RSF of 13 and 28 acres per hour in ice with an RSF of 10.

Because LACV-30 testing emphasized operational evaluation and comparative icebreaking techniques, it is much more difficult to quantify its actual rate of icebreaking. For example, values of 17 acres per hour in an RSF ice field of 23 were recorded during the Kankakee River operation, but these figures should be used with caution. The craft's effectiveness was hampered throughout the entire operation by a malfunctioning propeller pitch control which considerably reduced its maneuvering capability.

The use of ice resistance as a determining value of effectiveness was first employed by the Canadians in their ACV icebreaking studies (Reference 11), and a figure in their report, showing a resistance factor

MOTES	Channel closed in immediately after passing.		The consolidated brash 6 to 8 inches thick. The channel cleared was 1005, however, it closed back in within 100 yards of the passing. This test was run on only the keel engine with the other two idling.	Run 3 - Estimated RSF ice 30 Inches thick, some ice cracking was visible 50 yards either side of broken channel.	This data point represents the SUMAC breaking ice alone. Ice closed within one ship length behind boat.	Run 1 - This ice field was unbroken uniform ice 10+ inches thick.	Run 3 - This ice field was unbroken uniform ice 13+ inches thick. This test was terminated because of a bow down due to a blown segment.	Run 2 - Brash ice, some areas raffed up by tows 3 feet high, consolidated.	Run 3 - Brash consolidated field 8 inches thick, some raffing, SUMAC prop wash cleared channel about 100 feet wide, however, it would close in again = 150 feet behind.	Run 2 - Brash unconsolidated with occasional ice plates 6 to 8 inches thick.	Run 4 - Meavy brash unconsolidated and consolidated areas. Slow speed was required to prevent plow in of ACV bow.	Run 1 - Plate ice 90% cover with new knitting ice 3 inches thick.		Run 1 - Solid uniform ice 15 to 18 inches thick. Entered ice from running start.	Run 2 - Same Ice conditions as T.C. 17, run 1, settled down to steady state conditions.	Run 1 - Tight consolidated uniform thickness ice. Run made with SIMMC only, no ACV.	Run 6 - Run conducted in same area as T.C. 23, run 1, but with ACV attached.	Run 3 - Run conducted without ACV, light ice conditions.		Run 1 - Consolidated brash having a tendency to pile up under cushion.	Run 7 - Heavy brash packing up under cushion, most likely due to SUMAC's bow, necessitated frequent stops to clear cushion
7.C.	003	900	500	800	600	010	010	10	012	013	110	910	910	110	110	023	023	023	023	024	024
RSF	-	6	æ æ	9	2	9	82	=	=	02	21	13	15	23	23	02	9	02	0	=	=
CHANNEL CHANNEL BROKEN	**	150	09	98	40	70	S	8	20	45	90	90	30	45	45	93	40	30	40	30	30
COVERED WITH ICE	100	Z. Z	I	98	:	95	S.	96	96	09	06	06	96	06	06	20	20	50	50	06	06
RATE*	22.7	16	18.2	33	20	15	23	50	18	18	=	6	6	24	4	24	12	92	28	12	20
SUMAC FUEL CONSUMP.	1.1	\$.09	0.6	69.4	111.7	15.5	¥.5	64.0	115.5	13.8	50.3		28.8		94.5	0.09	48.0	56.2		55.1	9.06
SPEED	-	-		3.2	4.2	8.	3.8	3.4	3.0	3.4	7.	2.4	5.6	4.5	3.1	6.9	9.6	5.6	5.7	3.4	9.6
H.P.	1069	1015	9	1223	1246	292	633	901	1955	240	857	255	200	1528	1682	1062	808	963	116	920	1559
	1044	-		796	154	594	806	686	1153	579	126	615	759	1050	1057	897	913	226	926	927	1109
M .	-	2	-	-	5	9	-	00	6	2	=	12	13	=	2	9	-	18	61	20	23

* Rate broken in acres per hour, 1 acre = 43,560 sq. ft.

By Acres where B is the breadth of the channel broken

8.25 * hr V is speed in mph

** Assumed to be width of ACV which is either 45 ft. or 55 ft. depending on which side was faced up to the SUMAC.

NR Mon-recorded, i.e., did not get recorded during test run.

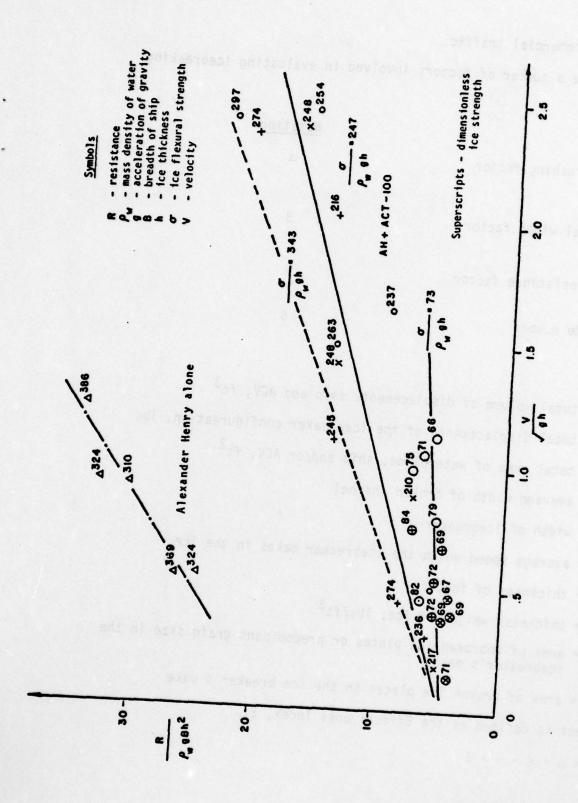
The T.C. Mos. do not run consecutively because some tests, such as stopping and maneuvering, were not intended to obtain data on rate of icebreaking.

RSF - Relative Strength Factor

TC - Test Control Number

vs the Froude number has been accepted as a standard method for demonstrating icebreaking effectiveness. This figure, reproduced as Figure 6-2, shows a plot of a non-dimensionalized resistance in ice factor, R/pgBh vs Froude number (based on ice thickness). An upper line on the figure shows the resistance of the push boat ALEXANDER HENRY without the ACV and a lower line is the resistance of the ship with the ACV. It should be noted, however, that lowered resistance alone is misleading as an indicator of system effectiveness. The resistance of the pusher vessel is quite naturally lower since the ice had already been broken by the ACV before it encountered the hull of the pusher vessel. Resistance encountered in pushing the ACV over the ice field is not comparable to that which the pusher vessel alone would encounter in breaking the ice pack. Also, the price for the lowered resistance is paid in terms of the capital cost of the ACV with its installed power together with its amortization, fuel, and maintenance costs. A more complete indicator of effectiveness should reflect the degree to which the total system accomplishes its total mission, which is to provide a navigable channel.

An outgrowth of this test and evaluation program has been the development of a theory for measuring icebreaking effectiveness in terms of the effect of broken ice on a non-icebreaking vessel following in the channel. It is now proposed that consideration be given to the development of a comprehensive model which will serve this purpose. During the course of the project, an initial attempt was made to lay the methodological groundwork for such a model and an <u>Effectiveness Index</u> was developed. It is intended to serve as a tool of quantification to provide a number which will indicate how well an ACV configuration is accomplishing its mission of making rivers



Dimensionless Resistance versus Dimensionless Velocity Figure 6-2

passable for commercial traffic.

There are a number of factors involved in evaluating icebreaking effectiveness:

<u>Factor</u>		Notation
$\frac{A_{i}}{A_{f}}$	Ice crushing factor	α
$\frac{B_C}{B}$	Channel width factor	В
tWice \(\Delta / Awp \)	Ice resistance factor	Y
V √ g∇ 1/3	Froude number	δ

Where ∇ = total volume of displacement, ship and ACV, ft³

 Δ = total displacements of the icebreaker configureation, lbs

 A_{wp} = total area of waterplane, ship and/or ACV, ft²

B = average width of broken channel

B = width of icebreaker

V = average speed which the icebreaker makes in the ice

t = thickness of ice, ft

Wice = thickness weight of ice, lbs/ft^3

A_i = area of unbroken ice plates or predominant grain size in the icebreaker's path

 A_f = area of broken ice plates in the ice breaker's wake The product is defined as the Effectivenss Index, E_I .

$$E_I = \alpha \cdot \beta \cdot \gamma \cdot \delta$$

By taking the product of these factors we are, in effect, placing equal importance on each. An alternative scheme which would permit weighting of cost factor would involve a weighted sum.

 $E_I = W_{\alpha} \cdot \alpha + W_{\beta} \cdot \beta + W_{\gamma} \cdot \gamma + W_{\delta} \cdot \delta$ where W_{α} , W_{β} , W_{γ} , W_{δ} are weighting factors assigned in order of the relative importance of each factor.

The Effectiveness Index, $\mathbf{E}_{\mathbf{I}}$, considers only the effectiveness with which an ACV configuration accomplishes its mission. Power was not introduced because power is not a cost item. It is reflected in the capital cost of the system, fuel costs and maintenance costs. The variable which we will define to characterize the power is:

$$P_{I} = \frac{\nabla \cdot \Lambda}{b}$$

where P = total power, including cushion power, of the system. Thus a higher value of $P_{\rm I}$ indicates a more favorable use of power.

A plot of P_I vs E_I for the various cases tested can be made to illustrate graphically the interrelationships of these variables and show the relative effectiveness of each icebreaker configuration in performing its mission. Such a plot was considered to be beyond the scope of this project.

6.10 Ice Management

It became apparent early in the test and evaluation program that effective icebreaking operations should involve the establishment of a system of ice management which breaks the ice in the channel but does not create new problems elsewhere by the presence of broken ice. It was observed, for example, that currents on the Illinois River and in Peoria Lake were not strong enough to flush broken ice downstream and that ice

disposal was as great a problem as icebreaking. Furthermore, floating chunks of broken ice interfere with lock and dam functions and are a potential source of damage to these and other river structures, such as bridge piers and boat docks.

In recognizing and understanding the complexity of ice problems, several factors were identified which require attention if a comprehensive ice management system is to be developed. Specifically, the system should provide:

- o Strategies for implementing induced flow through problem areas of the river
- Spillways to route ice chunks safely around lock and dam facilities
- Proper protection for hydraulic structures against ice induced damage
- Coordination of icebreaking efforts to ensure a passable river channel

A suitable ice management program would be advantageous in several ways. Close monitoring of conditions from the onset of the ice season would identify areas of ice buildup. Early and coordinated icebreaking efforts could control gorging and other complications. These efforts should help assure a navigable channel for towboats. A beneficial side effect would be reduction of spring flooding potential.

Associated closely with ice management is the control of river traffic in ice-clogged regions. During the project, it was observed that towboats can transit problem areas if they cooperate by using teamwork and a one-way traffic scheme. In such problem areas as Peoria Lake, one-way traffic regulation would prevent two towboats heading in opposite directions

from meeting in the middle of the ice-constricted channel. When a confrontation of this nature occurs, the passing operation forces the hardpacked ice at the edge of the channel back into the channel, with the result that navigation in that area is once again impeded. In the one-way system, towboat operators work in teams to push barges alternatively upstream and downstream across problem areas. Teamwork operations can further assure that barges which have to be abandoned are pushed far enough out of the main channel so as not to become obstacles to traffic. The one-way traffic and teamwork system was witnessed during the project, but not all towboat operators honored it. Its advantages should be obvious to all towboat operators, and the system could be managed by the industry itself without the necessity of government involvement.

6.11 <u>Icebreaking for Flood Control</u>

The operation of the LACV-30 in icebreaking for flood control contributed significantly to the understanding of ACV icebreaking technology. The operation resulted in a very successful effort and, according to the U.S. Army Corps of Engineers, Chicago District, was fundamental in averting possible serious flooding in the lower Kankakee River area.

This operation demonstrated some unique aspects of the icebreaking which had not been demonstrated previously. These are:

- Operating the ACV for extended periods away from the main base support facilities.
- o Operating in restricted areas with little room for maneuvering.

- Operations which required transit over land to go around bridge obstructions
- o Operations out of normal towboat channel
- o Operations in breaking up gorged ice conditions
- Operations requiring icebreaking starting from a hole cut in the ice for that purpose
- o Operations in proximity to populated residential areas
- Direct measurement of the relationship between wave patterns generated and icebreaking results
- Coordination of icebreaking with river flow control using the Dresden Island Dam settings.

Icebreaking for flood control was conducted on the Kankakee River from 1 March 1978 to 18 March 1978. The ice broken during this operation was approximately 20 inches thick and it was broken at rates up to 17 acres per hour.

6.12 Sound Levels

Whenever possible, sound levels were monitored for both craft using a sound level meter manufactured by the Bruel and Kjaer Instruments Co., Inc. (Type 213H). Readings were recorded on clear days with low wind conditions. The following section presents sound level information for both ACV's.

Sound levels on the RIVER GUARDIAN itself during operations ranged from 80 decibels (db) to 124 db depending on location. The 80 db reading was recorded inside the control cabin with the door closed while the readings of 124 db were recorded when the sound level meter was placed flush against the fan intake openings. However, five feet away from the openings of each fan intake the sound level dropped to 110 db and average deck level sound readings varied between 98 db and 108 db.

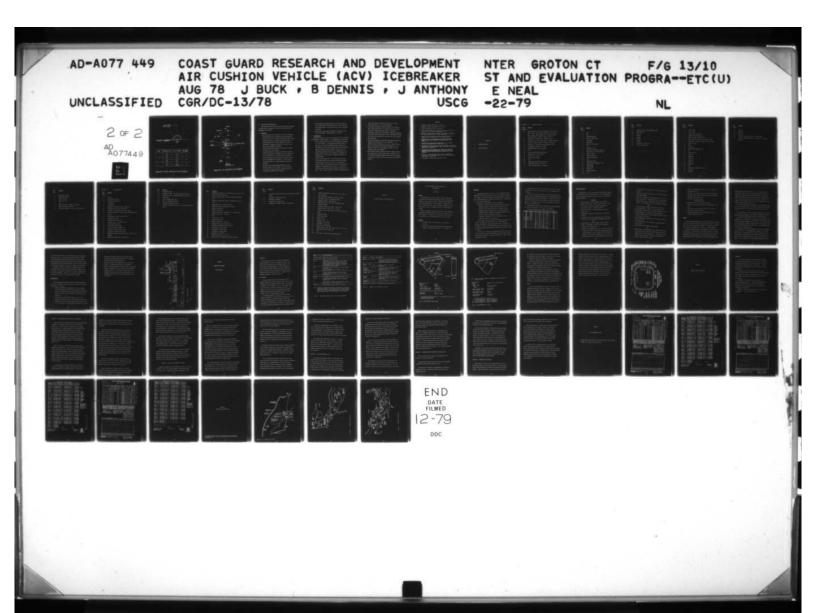
Readings on the SUMAC also varied widely. While pushing the RIVER GUARDIAN, readings of 62 to 64 db were recorded on the bridge with the doors closed, 96 to 104 db on the main deck, 76 to 80 db in the crew's quarters*, and 103 db in the engine room.*

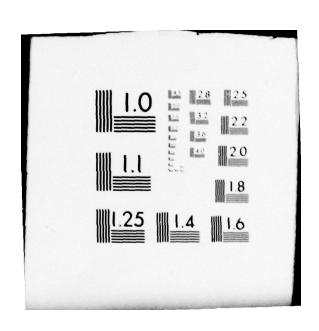
Figure 6-3 provides a diagram of sound level readings at various distances away from the SUMAC/ACV as it proceeded with its operations.

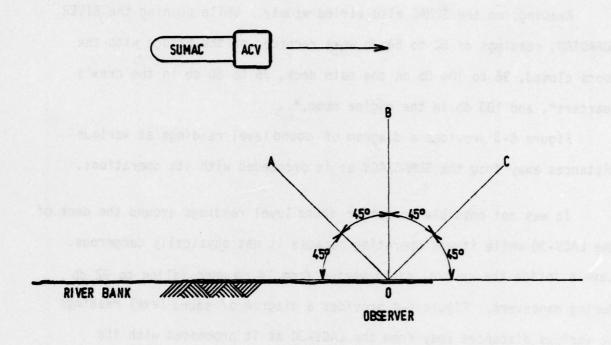
It was not possible to gather sound level readings around the deck of the LACV-30 while it was operating because it was physically dangerous. Levels inside the control cabin varied from 74 db when idling to 92 db during maneuvers. Figure 6-4 provides a diagram of sound level readings at various distances away from the LACV-30 as it proceeded with its operations on the Kankakee River. In addition to these readings, reports were registered that the craft could be heard up to eight miles away. It is interesting to note that in comparison, a 707 aircraft registered 111 db on take-off and a DC-9 aircraft registers 102 db (Reference 12).

Surprisingly enough, despite the high sound levels of the craft, there were no complaints from the residents during icebreaking operations for flood control on the Kankakee River. The 116 db readings were taken within 25 feet of the craft on the Will County Bridge where there were many spectators. Those residents were more interested in the operation's goal of flood prevention. It is unclear whether public reaction would be the same elsewhere.

^{*} Readings in the crew's quarters and the engine room were made with and without the ACV bow attached to the SUMAC. The crew's quarters were noisy because they were in the SUMAC's stern section right over the propellers. Ice could be heard hitting the propellers and engine whine also added to the sound level.







READING	DISTANCE FROM ACV(FEET)	NOISE LEVEL READING (db)	ACV LOCATION REL. TO BANK
1 101, and	200	74.5	450
A de mandi w	400	57.0	
1512992 47 B	150	7.0	900
ed pools	500	65.5	ne dánables
C	200	71.5	135°
os ruty si	600	5810 60	133

Figure 6-3. Sumac / ACV sound level diagram

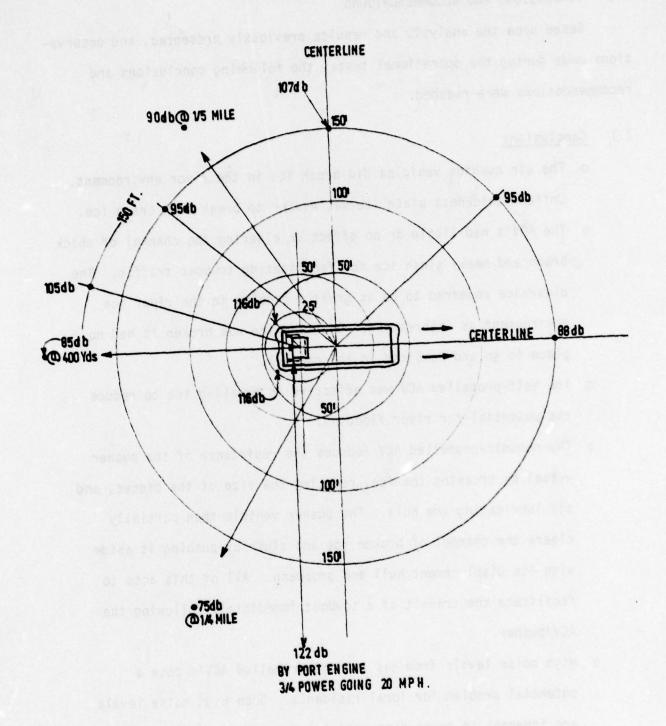


Figure 6-4. Lacv-30 sound level diagram

7.0 CONCLUSIONS AND RECOMMENDATIONS

Based upon the analysis and results previously presented, and observations made during the operational tests, the following conclusions and recommendations were reached.

7.1 Conclusions

- O The air cushion vehicles did break ice in the river environment.

 Uniform thickness plate ice was easier to break than brash ice.
- o The ACV's had little or no effect on clearing the channel of thick brash and heavy slush ice for facilitating towboat traffic. The clearance appeared to be as great a problem in the river ice environment as icebreaking. Once the ice was broken it had no place to go and remained in the channel.
- o The self-propelled ACV was effective in breaking ice to reduce the potential for river flooding.
- o The nonself-propelled ACV reduces the resistance of the pusher vessel by breaking the ice, reducing the size of the pieces, and air lubricating the hull. The pusher vehicle then partially clears the channel of broken ice and slush by pushing it aside with its displacement hull and propwash. All of this acts to facilitate the transit of a towboat immediately following the ACV/pusher.
- o High noise levels from gas turbine propelled ACV's pose a potential problem for local residents. Such high noise levels are inherent to turbo-prop propulsion systems but may be controlled by future designs of ACV propellers.

- o The Coast Guard can operate and maintain air cushion vehicles with only a minimal amount of additional personnel training. Operation of self-propelled ACV's is the area where most of the training would be required.
- o The 60 oz/sq yd single weave skirt material was inadequate for the river ice environment. Adequate material is available.

7.2 Recommendations

Based upon the analysis, results, and conclusions, as well as observations and experiences of the test personnel, it is recommended that the Coast Guard:

- o Perform a complete study of the Coast Guard icebreaking requirements for inland waterways. The following items should be included in the study: ice and traffic management schemes, conventional and ACV icebreakers, winter severity, clogged channel clearing techniques, commerce requirements and season extension on particular rivers.

 The outputs of the study should include: the requirements for icebreakers (number and type), costs and benefits of icebreakers and recommendation for the facilitation of commerce and potential for season extensions.
- o Evaluate a self-propelled ACV in a long-term multi-mission role.

 A self-propelled ACV should be acquired, configured and outfitted for other missions such as SAR and A to N in a long-term multi-mission evaluation. The vehicle should be used in roles which take advantage of its amphibious capability (such as in shallow water SAR) and its speed (such as servicing navigation aids which are far apart). Then, it should be deployed for specific icebreaking missions

- such as early opening of the Upper Mississippi River or flood control applications on a river like the Kankakee.
- o Investigate new techniques to better define the river ice environment. Study the applicability of classifying the river ice using the Relative Strength Factor (RSF) technique. Improve the methodology of comparing the effectiveness of icebreakers by developing an effectiveness index procedure. Both of these technical studies would advance the understanding of icebreaking and river ice environment technologies.
- o Consider the development and operation of barge-type ACV's which are compatible with Coast Guard WYTL or WYTM icebreakers. This combination should be operated in rivers, bays or lakes on the East Coast or Great Lakes where icebreaking requirements exist. The addition of the ACV will greatly augment the capability of the conventional Coast Guard icebreaker.
- o Keep the towing industry informed through formal Technology Transfer of the capabilities of nonself-propelled ACV's. When their reliability becomes sufficiently accepted, operators should be encouraged to push them at the head of their tows to solve their own icebreaking problems.

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APPENDIX A

- A.1 SUMAC/ACV OPERATIONS
- A.2 LACV-30 OPERATIONS

APPENDIX A.1 SUMAC/ACV OPERATIONS

Date	<u>Operation</u>
December	
1-11	Outfit SUMAC and ACV with R&D instrumentation. Operate out of
	Base St. Louis. Run open water checkout runs in Mississippi
	River. Resolve initial problems such as ACV diesel engine
	blower malfunction. Prepare for transit to Peoria.
12	Underway in p.m. Barge on hip. Overnight at Alton.
13	Ice gorge at mile 30.6, Illinois River. Overnight at Kampsville.
14	Heavy ice. Barge towed astern. Overnight at Beardstown.
15	Barge on hip. Arrive late at night, ANFAC, East Peoria
16	Arrangements for SUMAC while at ANFAC.
17	Moor SUMAC barge for winter operation
18	Off
19	ACV skirt inspection at Pekin boat ramp
20	Attempt ballast. Crane snowed out.
21	Ballast ACV. Trim level
22	Secure ACV for Christmas
23-28	Off
29	SUMAC alone open water test, Peoria Lake
30	SUMAC-ACV open water in Peoria Lake
31	ACV not hovering properly - no operations

Date	Operation
January	
1	Off
2	Off
3	Grade Beaching Ramp
4	Beach ACV, skirt change
5	Skirt change
6	Retrieve ACV
7	Confer on skirt problems
8	Off
9	Ballast to 1.1 PSI
10	Beach ACV, skirt change, retrieve ACV
11	Break clear ice, near Army Corps pier
12	Beach ACV, repair
13	Repair, retrieve ACV
14	Operate in Peoria Lake
15	Off
16	Grade ramp, beach ACV
17	Repair skirts, main engine oil leak
18	Repair engine
19	Repair engine
20	Retrieve ACV
21	Operate in Narrows
22	Off

Date	Operation
January	
23	Beach ACV, repair, operate McCluggage Bridge
24	SUMAC light boat
25	Beach ACV
26	On beach
27	On beach
28	On beach
29	On beach
30	On beach, skirt liners arrive
31	On beach, install liners

Date	Operation
February	
	•
1	Install liners
2	Install liners
3	Install liners, retrieve ACV
4	Operate 17 miles in Peoria Lake
5	Beach ACV, inspect, repair, retrieve
6	Assist towboats in Peoria Lake. Overnight IVY Club
7	Break clear ice, Convoy Ops, SUMAC & ACV limits reached
8	Beach ACV, repair
9	Repair
10	Retrieve ACV
11	Operate near McCluggage Bridge
12	Off
13	Beach ACV, repair
14	Repair ACV
15	Repair ACV
16	Repair ACV
17	Repair ACV
18	Repair ACV
19	Repair ACV
20	Off
21	Retrieve ACV, operate Detwiler light
22	Beach ACV
23	Repair ACV

Date	Operation
February	
24	Repair ACV
25	Repair ACV
26	Repair ACV
27	Retrieve ACV, deploy to Havana and overnight there
28	Operate to LaGrange Lock and Dam. Towboat assists. Beardstown
	overnight

<u>Date</u> March	<u>Operation</u>
1	Transit back to ANFAC
2	SUMAC rupture fuel tank
3	Clean up oil spill
4	Clean up oil spill
5	Off
6	Beach ACV, repair; one segment, retrieve ACV
7	Operate with towboat Jesse Brent
8	Return ACV to Mackley Ace, Inc. at Peoria Lock and Dam

APPENDIX A.2 LACV-30 OPERATIONS

Date	Operation
January	
1-2	Holiday.
3-5	Crew returns from leave
6	Cold weather modification
7	No operations
8	No operations
9	Maintenance, operate Peoria Lake, 2 engines down
10	Maintenance fuel system and rudder cable
11	Maintenance auxiliary power unit and port engines
12	Maintenance lift fan and #1 engine, Bell Canada Representative
	arrives
13	Maintenance pitch control and port engine
14	Maintenance port engine
15	Maintenance port engine. Run up
16	Maintenance skirt. Fuel control problem
17	New fuel control installed
18	Identify fuel control problem
19	Fuel leak #2 engine. Run fuel line
20	Maintenance #2 engine and fuel system
21	Maintenance #2 engine and fuel system
22	Maintenance #2 engine. Operate Peoria Lake. Starboard pitch
	casualty
23	Maintenance starboard dome. Defroster prod.

Date	Operation
24	Dome repair attempt
25	Maintenance routine. Bell Canada Representative departs
26	Maintenance, routine. Replacement dome delayed by weather
27	Awaiting dome
28	Install starboard dome. Run up
29	Run up. Starboard pitch problem
30	Disassemble starboard pitch control
31	Reassemble starboard pitch control. Bell Canada Representative
	arrives

bannadin – andaranja da

Date	Operation
1	Maintenance starboard side. Operate Peoria Lake. Towboat assist
2	Ballast to replace swing crane
3	Operate Lower Peoria Lake. Maintenace lift fan and starboard
	pitch pitch
4	Maintenance lift fan and #1 engine. Starboard side vibration
5	Off
6	Operate Peoria Lake. Repair skirt system
7	Replaced and balanced both propellers and domes
8	Operate in Peoria Lake
9	Operate in Peoria Lake
10	Operate in Peoria Lake. Towboat assist. Test magnetometer
11	Operate in Peoria Lake. Ice Survey
12	Off
13	No operations - blizzard
14	Operate in Peoria Lake
15	Operate in Peoria Lake. Ice Survey
16	Round trip to mile 100 of Illinois River
17	Round trip to LaGrange Lock and Dam (mile 80)
18	No operations - holiday routine
19	No operations - holiday routine
20	No operations - holiday routine
21	Maintenance. Skirt repair
22	Operate in Peoria Lake. Ice survey
23	Transit to mile 56. Overnight at LaGrange Lock and Dam

Date	Operation
February	
24	Transit to Alton (Mississippi River) and return to and overnight
	at Largo
25	Transit back to Peoria Lock and Dam
26	Maintenance. Scheduled.
27	Maintenance skirt (Lifted craft)
28	Maintenance. Bow fingers replaced. Keep bag repair

Date Operation March Transit to Kankakee River (Illinois, mile 270). Break one mile of Kankakee 2 Break second mile of Kankakee (to County line bridge) 3 Maintenance, general. Port propeller rigging Maintenance, general. Snow ramp built 5 Rebreak lower river. Cross road. Maintenance, general Maintenance, general 7 Maintenance, Replace port propeller. Skirt repair Operate by bridge. Chain saw ice. No success Crane break ice by bridge. Operate in afternoon. Small area broken Operate near bridge 10 11 Operate above bridge 12 Operate in river 13 Operate in river Maintenance. Skirt repair 14 15 Maintenance. Skirt repair 16 Operate up to railroad bridge 17 Operate in river. Cross road. Rebreak lower river. Ice flows Transit back to Peoria Lock and Dam 18 Off 19 20-on Test secured. Awaiting word to disassemble LACV-30

APPENDIX B

HULL SURVEY REPORT OF USCG SUMAC (WLR-311)

HULL SURVEY REPORT OF USCGC SUMAC (WLR-311) 13/14 MARCH 1978

by

JON R. BUCK

GENERAL

The USCGC SUMAC was used as the tow boat to push the experimental air cushion vehicle icebreaker RIVER GUARDIAN during test and evaluation through the 1977-1978 winter ice season. The SUMAC started operations with the RIVER GUARDIAN at St. Louis on December 1, conducting tests on the ACV icebreaker from that time until testing was terminated at Peoria Lock and Dam on 9 March. The SUMAC returned to St. Louis arriving on 11 March.

This Hull Survey Report presents the results of the hull inspection to determine what damage was caused to the vessel as a result of the ice-breaking operation. The inspection was conducted on 13 and 14 March at St. Louis Ship Yards, 611 East Marceare Street, St. Louis, Missouri.

REFERENCES

- (a) CGC SUMAC, Book!et of General Plans, 8/12/70, DRW No. 179 SP-0103-4.
- (b) Record of the Ship Structural Evaluation Board for USCGC SUMAC (WLR-311), Second Coast Guard District, 3 - 6 October 1977.

BACKGROUND

It was determined before the testing of the ACV icebreaker that the SUMAC would require some modification to the hull to minimize the potential of ice damage. The boat was dry docked at Mainstream Shipyard and Supply, Inc., Greenville, Missouri, to complete these modifications. The following is a list of these modifications:

Modifications to USCGC SUMAC for Icebreaking Operations

- Installed ice fence deflectors around Skin Cooler 3/8 inch plate inboard and outboard 9 inches high between frames 24 and 45.
- Installed a "V" shaped 3/8 inch plate ice deflector on keel between and ahead of frames 22 to 24 skin coolers (small "V").
- Installed a large "V" shaped 3/8 inch plate ice deflector from keel out to knuckle from frame 48 to frame 56.
- Installed 6 ice knives of 2" x 14" x 30" plate at frame 54.
 NOTE: See reference (a) for locations.

The ice fences, Item 1, were installed to afford some protection to the skin cooler which protruded below the hull. These fences were installed longitudinally along the sides of the cooler. The small "V" ice deflector, Item 2, was installed between the port and starboard cooler to prevent ice chunks from being caught or jammed into the keel area between the coolers.

The large "V" ice deflector was designed and installed to sweep the ice outward and away from the flanking rudder, props and steering rudder. Both the small and large "V" deflectors have holes cut in them to minimize added drag to the boat.

The installation of the large ice knives, Item 4, was also intended to minimize the potential of ice hitting the critical appendage in the stern area.

During the dry docking at Greenville a thorough hull survey was conducted. Hull plate thickness was measured and it was determined that some locations had as much as 54.7 percent reduction. It was decided to reinforce these locations because of the potential of damage in icebreaking operations. The addition of a 3/8 inch plate was welded over the hull in the area of bow frames 10 to 18.

A representative sample of the critical plate thickness measurements are included in this report. (See reference B for details.)

Critical Plate Thickness Where Reinforcement was Required

Location	Strake	Frame	Design	Test	Red. In.	% Reduction
Port	Keel	17	.375	.170	.205	54.7
Port	Keel	17	.375	. 285	.09	24.0
Port	Keel	21	.375	.29	.085	22.7
Port	Keel	23	.375	. 24	.135	36.0
Port	Keel	24	.375	. 24	.135	36.0
Starboard	Keel	134	.375	.275	.10	26.7
Starboard	Keel	17	.375	.195	1 .18	48.0
Starboard	Keel	231	.375	.270	.105	28.0
Starboard	Keel	25	.375	. 265	.11	29.3
Port	A	161	.375	.265	.11	29.3
Port	A	175	.375	. 25	.125	38.3
Port	A	181	.375	. 265	.110	29.0
Port	A	191	.375	.235	1.14	37.3
Port	A	2012	.375	. 245	1 .13	37.7
Port	A	21	.375	.245	.13	37.7
Port	A	214	.375	. 280	.095	25.3
Port	A	2212	.375	.380	.095	25.3

HULL SURVEY RESULTS

The following is a listing of the damage to the SUMAC which appeared to be the result of the test and evaluation of the ACV icebreaker operation. The results are presented by areas of the hull, i.e., Starboard, Port, Stern and Bow.

STARBOARD

- 1. Scoop valve guard severely damaged -- must be replaced.
- Large "V" ice deflector bent all along its length. Broken off at knuckle. Center point crushed in against hull.
- 3. Skin cooler ice fence only slightly dented.
- Small "V" ice deflector severely dented and crushed at center point.
- Slight bend in inboard pipe of skin cooler at a point about mid-ship.
- Slight dent in fairing on leading edge of skin cooler (could have been previously damaged before icebreaking operations).

PORT

- 1. Scoop valve guard severely damaged -- must be replaced.
- Large "V" ice deflector moderately bent along length.
 Severely bent on outboard side.
- 3. Fracture of fuel compartment (Port aft) at a point 3 inches along the weld of the large "V" ice deflector, approximately 18 inches in from the knuckle (after the first cutout hole of the ice deflector). The fracture is approximately 1/2 inch long. This fracture appeared to be in the heat sensitive zone of the weld

- and was caused by the tearing away of the ice deflector in that location.
- Slight denting of fairing of the skin cooler near keel (possibly from previous damage).
- 5. Dent in hull plating and stiffners between frames 2 3 and 3 4 above waterline about 1 foot below deck level. No visible paint scratches on outside but flaked paint on inside indicating a new dent. That compartment had been newly painted at Greenville.

STERN

- 1. Port Flanking Rudder missing, broken off flush with hull.
- 2. Keel Flanking Rudder missing, broken off flush with hull.
- 3. Keel Propeller slight dents in two blades, repaired in place.
- 4. Starboard Propeller severely dented -- must be replaced.
- Starboard Steering Rudder missing, rudder post broken off about
 inches below hull bearing.
- Keel Steering Rudder rudder post bent to port about 4 degrees bend at hull point.
- Port Steering Rudder rudder post bent to starboard slightly about 1 degree.
- 8. Port and keel rudder restraining chains missing.

BOW

1. Starboard Knee - rubber damaged at bottom, steel retaining bracket ripped off on both sides up from the bottom for about 3 feet on inboard side and 1 foot on outboard side. Steel retaining bracket dented at several locations above water line (probably due to collision with ACV at Pekin, Illinois.

- Keel Knee rubber damaged slightly at bottom. Both sides of retaining bracket bent.
- 3. Port Knee steel retaining bracket on outboard side missing for about 8 feet up from bottom. Inboard side torn free and rolled back about 3 feet up from bottom.
- 4. Four anodic zincs missing on bow.

GENERAL OVERALL VIEW

- 1. Bow and bottom paint scoured down to bright metal.
- Waterline region scoured paint down to bright metal about 1 foot wide.
- 3. Paint scoured to a slight degree from knuckle to waterline.
- 4. Stern area has some paint scouring.
- Occasional slight dents in hull not necessarily due to ice-could have been there previously.

SUMMARY

There was considerable damage to the ice deflector "V"s both the small one in front of the skin cooler and the large one in the stern. It is doubtful if the fuel tank would have been damaged if the ice deflector had not been torn away. The large 2" x 14" x 30" ice keepes were undamaged.

The steering rudders were most likely damaged in turning and backing down maneuvers. It appears that when the SUMAC instituted a turn in heavy ice that large plates of ice would slide under the hull and place a large athwartships load on the rudder post. As indicated by the port rudder post being bent inboard to starboard and the keel rudder unprotected by the

missing starboard rudder being bent to port. It appears that loads in turns might be greater than loads from backing down provided that the rudders are centered during the backing down maneuver. The starboard rudder had been broken off three times during the ACV icebreaking operations. One time was definitely during a turn to port in heavy ice (18 inches of uniform thickness plate ice) causing the starboard stern of the SUMAC to slide up against the ice field.

The starboard and keel propellers were most likely damaged when they hit the loose starboard rudder. This rudder was restrained by the rudder chain and was pushed into the propeller when the boat was backing down in ice.

It is interesting to note that the ice fences longitudinally along the skin cooler were not damaged. It is difficult to determine if the forward small "V" afforded any additional protection to the skin coolers.

The cause of the dent in the hull on the port forward location is undetermined. The crew members of the SUMAC were interviewed and the time and event causing this damage could not be verified. On one occasion when the tow boat THOMAS was taking on portable water at ANFAC, Peoria, it ran into the SUMAC which was moored at the facility. It is possible that this dent could have occurred at this time.

The damage to the towing knees appeared to be caused by ice between the ACV and the boat. As the ACV/SUMAC moves through an ice field the large plates of ice are pushed down by the cushion pressure, they pass under the craft and emerge again from the cushion near the knees of the SUMAC. However, the SUMAC had operated in heavy ice without the ACV on its bow. It is understood that the SUMAC encountered heavy ice on its

return trip to St. Louis and on several occasions had to back and ram the ice to get through. It is possible that the major damage to the knees could have occurred during the non-ACV attached time. During periodic docking operations of the ACV the SUMAC's knees could be observed. During the icebreaking test and evaluation operation it was noted that some damage had occurred to the knees but, that it did not appear to be so extensive.

The damage to the scoop valve guards is understandable as they protrude from the side of the hull and are particularly vulnerable. They are, however, a minor item. The scouring of the paint was expected and is not greater than expected. The loss of the sacrificial anodic zinks was also expected as they are in a vulnerable area for ice impact.

RECOMMENDATIONS

The following items are included as possible recommended modifications to the hull of the SUMAC or other similar boats to be used in ACV icebreaking operations in the future:

- Install ice deflector posts approximately 4 inches in diameter,
 feet long, adjacent but outboard to the steering and flanking
 rudder posts. Each post should be gussited to reduce stress concentration. This post will help to protect the rudders during turns in heavy ice.
- Install, where clearance permits, gussets at the top of each rudder post to distribute the load and reduce stress concentrations.
- 3. Install more ice knives to protect rudders and skin coolers in lieu of "V" ice deflectors. The knives could be positioned in a "V" shape

- along the hull to assist in moving the ice chunks outboard.

 This technique reduces appendage drag over the "V" deflector now used but provides protection from ice.
- 4. Beef up the fairing to the leading edge of the skin coolers.
 The small "V" between the coolers need not be installed.
- 5. Place a bow modifier between the SUMAC and the ACV. This modifier would act to displace the ice to the side of the boat instead of pushing in beneath it. The bow would act like a plow. It is recognized that an ice bow was used on the SUMAC before. The recommended modified bow for ACV operation would not be of such size and bulk.

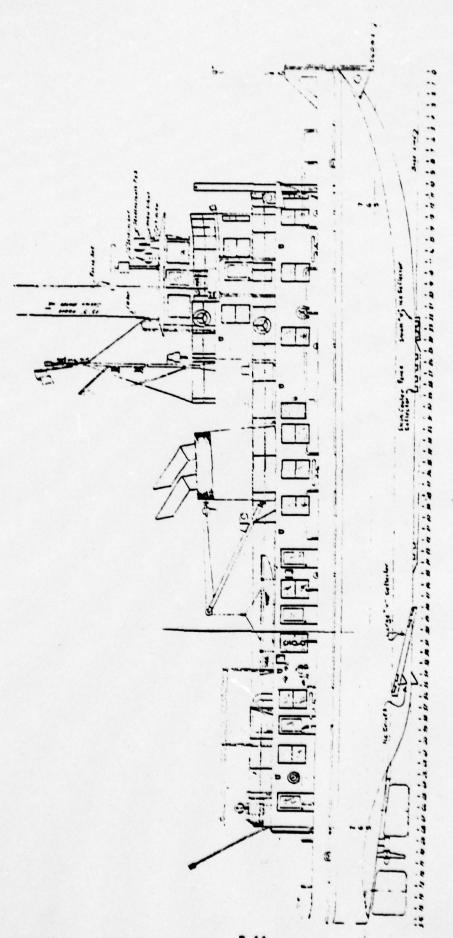


Figure B-1 SUMAC Hull Diagram

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APPENDIX C

SEGMENT REPAIR SUMMARY

RIVER GUARDIAN

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INTRODUCTION

During the project the skirt system of the RIVER GUARDIAN suffered repeated failures due to its segments being torn. This section presents the specifications for segment material and the liners that were eventually employed to reduce the failure rate. It also examines the segment damage that occurred throughout the project. Major events that occurred concerning the skirt system during the project are listed in Table C-1. A summary of skirt maintenance procedures is provided in Table C-2.

THE SEGMENTS

The skirt system was made up of 96 individual cone-like segments suspended from 26 steel frame trays which could be slid out from under the craft for segment inspection, removal and replacement. Two segment sizes were employed. One size fit the straight frames while the other size was used on the curved corner frames. Both segment sizes, however, were of the same design with only minor dimension changes necessary to accommodate the frame variation. Figures C-1 and C-2 provide diagrams of a segment.

The segments were made of a flexible 60 oz/sq yd material consisting of rubber with an inner base type nylon weave. Due to the harsh ice environment, the segments suffered repeated damage necessitating frequent repairs. In an effort to reduce segment failure and insure a more reliable cushion for the ACV, a 24 oz/sq yd liner of nylon weave reinforced polyvinylchloride (PCV) was introduced in the

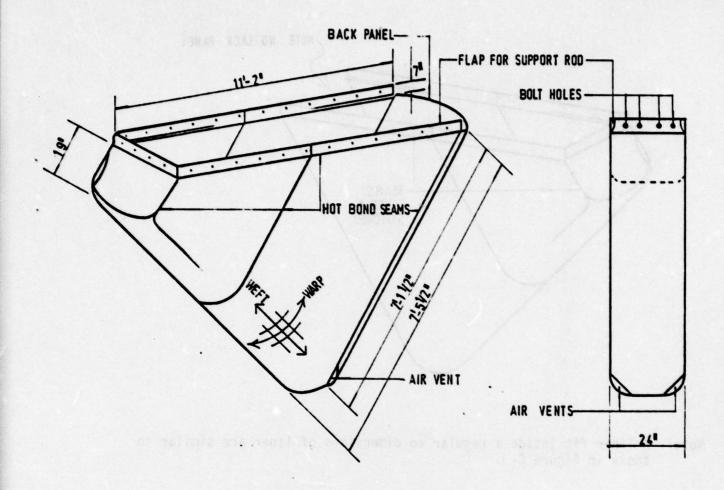
DATE	EVENT OR OBSERVATION						
12-12-77 - 1-5-78	Craft's "B" side served as bow						
1-6-78 - 3-8-78	Craft's "C" side served as bow						
12-19-77	Craft beached at Pekin, Illinois. Skirt inspection provided first evidence of segment damage. Back panels on segments were found ripped. Tears began at the base of individual segments and continued up the back panels adjacent to their hot bond seams. Not considered seriously damaged.						
2-3-78	Tire fender system installed over bow of ACV covering trays C1, C2, C3. PVC liners installed inside segments of trays +C, C-1, C-2, C-3, and -D.						
2-8-78	-C tray frame discovered bent between -Cl and -C2 segments. Overnight mooring in icefield thought to be responsible.						
2-13-78	PVC liners installed inside segments of trays -C & +D						
2-21-78 Tire fender system extended to cover trays +C and External protection patches installed on segments trays +C and -D. By this date all segment back panels had been remfrom trays B4, -C, +C, C1, C2, C3, -D, +D							
3-8-78	Final beaching of ACV at Peoria Lock & Dam site						

Note: Back panels on segments were designed to prevent skirt system from scooping up ice chunks. They were not considered of major importance in the skirt system and so their ripping was not considered serious. Later in the project, the back panels were cut out of many segments because they were ripped. They became useful as repair patches.

Table C-1 Major Events Concerning Skirt System on RIVER GUARDIAN

EVENT	TIME	MEN	COMMENTS
Beaching of ACV 45 min		6	This is an average beaching operation considering the repair pad to be adequate, but location and support equipment not necessarily designed for this purpose.
Removal of Spray Skirt	5 min	3	Removal of only 1/4 of the total spray skirt
Pulling of Straight Segment Tray	5 min	3	Pulling of straight side segment tray for inspection using a winch truck or Caterpillar tractor
Pulling of Corner Segment Tray	10 min	4	Pulling of corner segment tray for inspection. Requires 55 gal. oil drums to support the weight of the segment tray. Winch truck required.
Removal of Segment	15 min	4	Removal of a segment from one of the trays. This includes removal of the reinforcing steel straps.
Installation of Segment	15 min	4	Install new segment - bolted in place
Recouple Tray	10 min	4	Push tray back in place and secure

Table C-2 Summary of Skirt Maintenance Procedure



Segment Statistics*

Material Rubber Color Black

Material Weight 60 oz/sq yd

Tensile Strength (Warp) 450 lbs/sq in Tensile Strength (Weft) 350 lbs/sq in

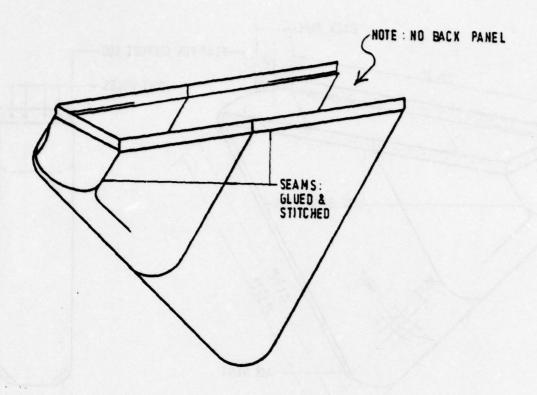
Tear Strength-Birds Wing Test**901bs (Warp & Weft)

Base Weave Nylon
Manufacturer Avon Co., England

* As supplied by Mackley Ace (USA), Inc. - See Reference 9 for results of tests run by VPI University

** British Standard Tear Test

Figure C-1 RIVER GUARDIAN Segment Diagram



Note: A liner fit inside a regular so dimensions of liner are similar to those in Figure C- I.

Liner Statistics*	
Material	"Polymit 19" PVC coated polyester
Color	Red
Material Weight	
Base	9 oz/sq yd polyester
Finished	23/24 oz/sq ud
Tensile Strength**(Warp)	800+ lbs/sq in
Tensile Strength (Weft)	800+ 1bs/sq in
Tear Strength***(Warp)	185 lbs
Tear Strength (Weft)	250 lbs
Coating Adhesion****	20 lbs/sq in
Cold Crack****	-27/-30°C

* As supplied by Mackley Ace (USA), Inc.

** British Standard Test - BS3424, 1961 Method 6A

*** British Standard Test - BS3424, 1961 Method 7A

**** British Standard Test - BS3424, 1961 Method 9A

**** British Standard Test - BS3424, 1961 Method 10

Figure C-2 RIVER GUARDIAN Liner Diagram

bow. These liners were of similar design to the rubber segments and fit conveniently inside 21 bow segments. Figures C-1 and C-2 provide the specifications for both the rubber and PVC segments.

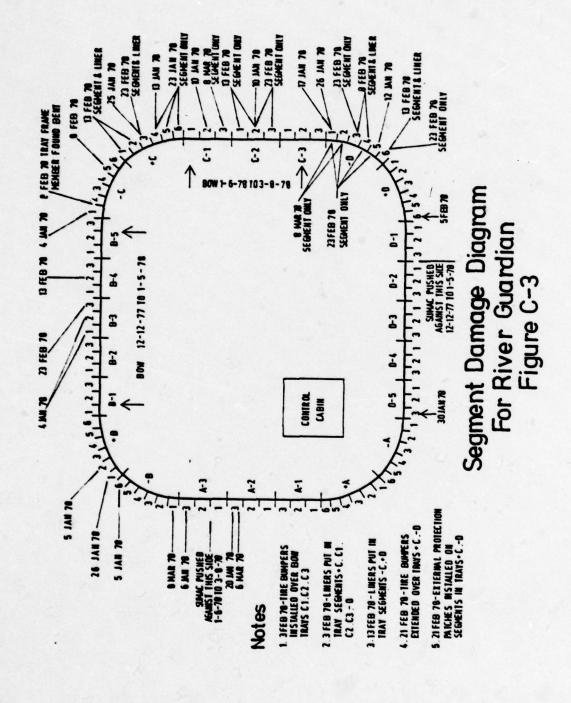
Originally, the entire skirt system of the craft was covered from view with a lighter-weight spray skirt made of nylon reinforced rubber. Its purpose was to prevent water spray created by the craft from spilling out and freezing on the deck of the ACV during operations. Unfortunately, the ice quickly destroyed this skirt.

At one point in the project, it was suggested that something should be provided to prevent sharp ice from puncturing the segments. A fendering system was, therefore, fabricated out of old automobile tires and suspended over the bow of the ACV. Whether the tire fendering system-directly helped to prevent or reduce segment failure was difficult to ascertain. The tire system was implemented at the time the liners were introduced but, even with these modifications, the segments failed.

REPAIR

The 60 oz/sq yd segment material proved to be no match for the ice conditions encountered on Peoria Lake. Segments, mainly in the bow region, were torn severely and required replacement. The rate of segment failure was totally unanticipated and, therefore, the supply of spare segments was quickly exhausted. To order and manufacture additional segments would have delayed operations until spring; therefore, Mackley Ace personnel set about the task of repairing the damaged segments in the field. Extremely damaged segments were cut up to provide patches for repairable segments. Pop rivets were originally used to secure

patches and mend minor tears. However, these soon ripped out. Eventually, patches were glued on and then stitched by a commercial company. This produced a stronger repair job than the pop rivet method, but a good number of segments repaired in this manner failed also. Finally, the PVC liners were placed in the bow segments, and outside segments in +C and -D trays were reinforced with a pop riveted patch along their most vulnerable front area. During subsequent operations, the outer segments on the bow failed with their reinforcing patches being torn off, thus creating holes from which tears developed. The liners, on the other hand, maintained their shape, thus enabling the craft to continue operations and successfully return to port. Very few liners failed. Figure C-3 provides a detailed summary of segment damage dates.



APPENDIX D

ANALYSIS OF PROJECT OBJECTIVES

INTRODUCTION

It is much more desirable to make direct measurements during a test and evaluation program than to collect opinions. Numbers can be interpreted much more clearly and further processed to yield and justify more far reaching conclusions. The problem is to identify what quantities can be measured and what is to be done with the data after the test.

The objectives of the test and evaluation are not directly measurable as they are stated. Each objective must be carefully analyzed and considered, then reduced to a measurable objective. It is these newly "reduced" objectives that will be measured during the test and evaluation.

The danger of this approach is that the essence of the original objectives may be lost. The following paragraphs address each of the objectives from the Test Plan for ACV Icebreaking and present the logic that results in the test measurements and individual tests (Reference 8).

Though this logic was originally employed for the barge-type ACV, its essence holds for the self-propelled ACV also.

Marine commerce may be broken down quite clearly into many elementary components; e.g., tonnage, cargo type, cargo value, transit time and so forth. One of the uses of these types of data is to assess economic impact, i.e., numbers are generally put into economic models that generate numbers describing revenue, rate of return, etc. It is far beyond the scope of the test and evaluation to assess the economic impact upon the towboat industry or the State of Illinois of the ACV icebreaking capability of the Coast Guard. But, it is desirable to obtain information on how ACV icebreaking affects the elementary components of marine commerce in the Illinois/Mississippi River System.

Ideally, the effect of ACV icebreaking upon commerce would be measured in the following way: over a given ice period, two identical Illinois River systems would be required, both having the same ice condition and towboats operating. The ACV icebreaker would operate in one of the two systems. Then all of the numbers describing commerce (number of towboats each day, tons on each tow, transit time, etc.) would be collected in each system. They would be compared and the difference would be due to the ACV icebreaker, the only difference in the two systems.

The above approach works well with models or with full-scale systems where duplicates are available. The Illinois River and its attendant commerce are not so easily duplicated. In lieu of this

impossible, ideal approach, several approximate methods will be used. They all will, in some way, attempt to emulate the results of the ideal approach but will fall short of the ideal due to their inherent limitations.

The first approach which approximates the ideal is to go back to the historical data from past years kept by the Army Corps of Engineers. Attempt to find the same weather conditions as this year over the same period of operation. In addition, establish that it was attempted to move the same amount of commerce. Then, compare the measure of commerce for this year (tonnage, transit time, etc.), with the ACV icebreaker present, and the historical year. The difficulties with the approach are numerous; e.g., it is difficult to repeat the same weather patterns, the economic health of the country dictates the potential barge traffic and the economy has been erratic recently, but barge traffic has been increasing. Even with these problems, a comparison can be made between the two sets of statistics and the differences between them attributed directly or indirectly to the ACV icebreaking.

The Army Corps of Engineers compiles voluminous statistics on the Illinois Waterway (as well as others). Reaching back into their data may provide a further means of comparison of like winters even though the daily profiles of temperature are not the same.

It may prove necessary to manipulate the data to obtain the measures of commerce that are demanded. Certainly unique or different quantities such as degree-days* of icebreaking could be generated from past data and capability to do so is at hand.

D-4

Another experimental approach is to intentionally leave a given reach of river unbroken (say five miles). Let a commercial towboat transit this reach and measure the time to transit and the fuel comsumed during the transit. Next, let the towboat transit a broken (by the ACV icebreaker) reach of the same distance and make the same time and fuel measurements. The differences in fuel consumption and transit time are then directly attributable to the air cushion vehicle icebreaker (the ice conditions being equal over the two reaches of river).

A more subjective approach is to interview experienced towboat captains on the river and ask them to assess the effect of ACV icebreaking upon measure of commerce, transit time, tonnage, etc. In addition, the ice damage suffered could be compared to a like winter without ACV icebreaking capability. This approach can provide the flavor of success or failure but provides no measurable quantity.

All of these approaches, as can be seen, fall short of measuring directly the effects of ACV icebreaking upon commerce, but, in the aggregate, the three approaches (1) finding a comparable year, (2) running the towboat with and without icebreaking, and (3) interviewing experienced towboat operators will supply a reasonable measure of the effect of the icebreaking upon commerce.

OBJECTIVE 2 - DEVELOP ACV ICEBREAKING PROCEDURES

Icebreaking procedures can be broken down into tactics and strategy.

An example of tactic is the speed and number of passes used to break

the ice near a dam. The method of deployment, responding to a boat

stuck in the ice or continually breaking the channel, would be a strategic decision.

The guide presently available to assist in developing techniques to break ice is the experience of the Canadian Coast Guard and the Icebreaking Annex (X) of the Second District OPLAN. The area of employment, (river versus open water), the requirements of ice breaking, and the great physical differences between the Canadian Craft and the SUMAC-ACV Bow make it necessary to start at a basic level.

Consider some of the differences in hardware and requirements.

The Air Cushion Vehicle bow pushed by the Canadian Coast Guard Icebreaker ALEXANDER HENRY was less than one tenth (1/10) the displacement of the ALEXANDER HENRY. The CGC SUMAC is closer to one fifth (1/5) of displacement of the Air Cushion Vehicle bow that she will push.

The Canadian icebreaker is a deep draft vessel with screws well below the water surface unlike SUMAC which is more flat bottomed and shallow draft and hence more subject to ice damage. In addition the Canadians had to break a channel in clear ice where the requirement on a river is to break the channel with all of the problem of bends, obstructions and refreezing after the towboats have passed through.

The increasing experience of both the Air Cushion Vehicle contractor and the Coast Guard operating personnel will enhance the development of icebreaking tactics. Some of the items to be considered are the number of passes in a given area, speed, jockeying controls, degree of control, and limits of all of the above. It is not apparent at

this writing what many of these limits are or what the behavior of the vessel is going to be in ice. The test runs will be very flexible so that procedures that show promise can be further explored and those that don't will not be continued.

The initial familiarization trails with SUMAC and ACV bow will provide a great deal of information on controlability, speed and other limiting factors. At that time those parameters can then be bounded in the test series. The initial ice operation will again provide further information which will narrow the procedures to be attempted. This will be done during the operating period as soon as practical.

Ultimately, probably more important than the tactics or procedures used to physically break the ice is the strategy of deployment and usage of the Air Cushion Vehicle icebreaker. Icebreaking as viewed in the past by the Second Coast Guard District has been in a response role, much, like SAR. If a towboat was stuck in the ice, the Coast Guard would assist him.

The Air Cushion Vehicle icebreaker offers the chance to keep a much larger area of the river open to traffic if used on a continuing basis along with a coordinated effort of the towboat operators.

If too many miles of river are "bitten off", then the icebreaking will be ineffective when the river refreezes. The convoy approach of escorting several towboats up or down the river is another method

of deployment that should be considered. This may be the only way to handle an area that gorges like Peoria Lake.

A final area of consideration is an ice clearance. The ice that is broken must either be pushed aside or further downstream. If it is pushed aside, it may ultimately lead to gorging in a narrow or restricted channel. If carried downstream, ice jams may occur at locks and dams. This is especially critical at dams where the possibility of losing the whole river pool exists if the dam is breached.

The strategic choices depend upon the weather. If the winter is mild, a large segment of the river can be broken on a regular basis by the Air Cushion Vehicle icebreaker. The towboats would then keep it clear with periodic transits. If the winter is more severe, assisting, escorting, or convoy techniques may be used. Commercial towboats must be used to clear dams in order that they not be breached.

OBJECTIVE 3 - COST EFFECTIVENESS OF ACV'S

It is necessary to define and bound cost effectiveness in the context of the test and evaluation. First, since the ACV platform is leased with an option to buy, it is not clear what the first cost is. The maintenance and life cycle costs are not presently known. Therefore, at this juncture, dollar cost figures are uncertain, but is still possible to establish relative and absolute cost figures for certain aspects of ACV icebreaking and these will be a good point

of departure for a future comparative costing study.

One of the most obvious benefits found by the Canadians in their icebreaking operation was the fuel savings in terms of gallons of fuel per square mile of ice broken. This quantity may be determined during the test and evaluation and be an input in the future for total operating cost of an ACV-pusher icebreaking combination. This data may then be compared with any conventional icebreaker to compare the fuel costs. The ACV icebreaker fuel consumption includes both the pusher vessel and the ACV.

Further, ACV costs may be assessed in the areas of damage and maintenance. All repairs, both man hours and replacement parts, are costs. These will be accumulated over the duration of the test and compiled. These data can then be used as input for life cycle costing where the life of a component must be considered. These data, again, are only a crude estimate since a very small portion of time and a single vehicle are going to be used, not a fleet which gives a much better statistical sample. The objective in determining costs is to measure what you can accurately and consider all other aspects for inclusion in a costing model.

The second part of cost-effectiveness is effectiveness. This may simply be measured as square miles of ice broken per hour. The quality of the broken ice is not a measurable quantity. It can be described in terms of size of ice pieces or the fraction of the channel

cleared and some more specialized ice descriptions. If the broken ice is in such large pieces that it requires a second pass to make the channel navigable, this effectively reduces the square miles of ice broken per hour.

Another complexity of effectiveness is the disposition of the broken ice. It ultimately must pile up on the side of the channel, flow downstream or melt. There may prove to be a limit to the effectiveness of icebreaking if gorging occurs. Further, the flushing of ice over the dams and through the locks is another limitation of the icebreaking operation. It follows that being too effective (breaking too many square miles per hour) can result in ice gorges and jams further downstream. Therefore, effectiveness is in terms of the rate at which ice is broken, but with the constraint that it may only reposition the problem further downstream.

OBJECTIVE 4 - DETERMINE TRAINING, PERSONNEL AND SUPPORT NEEDS

Each of the items in this objective are defined below in the context of this test and evaluation.

Personnel means the number, type and rate of the people necessary to operate and maintain the Air Cushion Vehicle bow.

Training means the additional training necessary for an MK (for example) to be able to satisfactorily maintain and operate the Air Cushion Vehicle bow.

Support needs are those goods and services that are not standard in the Coast Guard inventory that are necessary to adequately maintain and operate the Air Cushion Vehicle bow.

Observing the performance of the people operating and maintaining the ACV will provide the information necessary to establish the need for additional support and training. Self-propelled ACV's require operator training much like aircraft. The British Navy even has a school set up along these lines. Presently, it is assumed that the operating personnel will adapt readily to the nonself-propelled ACV, which is pushed like a barge. The operator is already familiar with the pusher vessel and the ACV bow is, therefore, the only unknown quantity. In the case of the self-propelled ACV, the operator must become familiar with the entire craft.

Maintenance of the ACV will be performed by Coast Guard personnel (when possible) supervised and trained by the ACV contractor. Their (CG) performance will reflect their skill and training. The need for non standard items such as crane service and a large beaching area will become apparent as they are needed.

OBJECTIVE 5 - DETERMINE DESIGN CRITERIA

Design criteria is taken to mean those numbers and curves which describe performance of the Air Cushion Vehicle. The basic description of air cushion icebreaking is the speed-ice thickness-cushion pressure envelope. These are influenced by power available and the platform

size. Further important engineering information includes noise levels, ACV roll and pitch attitude, ballast, and fuel consumption.

Actual operating data for a specific design provides an excellent point of departure for future design. Sizing the requirements of ice thickness determines the platform size and cushion pressure.

Actual tests data can be used to verify some of the empirical laws of ACV icebreaking design. The actual icebreaking mechanism has not been adequately modeled mathematically. Any information about the physical aspects of the breaking mechanism will be a valuable input for the future. Air cushion noise levels have been a point of contention in the past. Actual measurements are needed.

In addition to the ice breaking capability of the Air Cushion

Vehicle, the maneuvering characteristics are extremely important.

The Air Cushion Vehicle offers no directional stability and the combination of it with the pusher vessel is an unknown quantity. Standard maneuvering tests will allow a quantification of vessel performance in turns, stopping and passing. This will allow future vessels to be designed to, or exceed these particular values of maneuverability as required by the operators.

APPENDIX E

LOCAL CLIMATOLOGICAL DATA

(National Weather Service Office, Greater Peoria Airport, Monthly Reports.)
(December 1977, January 1978, February 1978)

DECEMBE# 1977 PECRIA. ILLINOIS NATIONAL MEATHER SERVICE OFC

GREATER PEORIA AIRPORT

Local Climatological Data

MONTHLY SUMMARY



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JANUARY 1978

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Samiel B. Mitchell DIRECTOR. NATIONAL CLIMATIC CENTER

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Local Climatological Data

MONTHLY SUMMARY

EBRUARY 1978

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APPENDIX F
KANKAKEE RIVER OPERATION MAP

This Appendix presents a map of the Kankakee River which provides an operational summary of LACV-30 icebreaking efforts for flood control in that area.

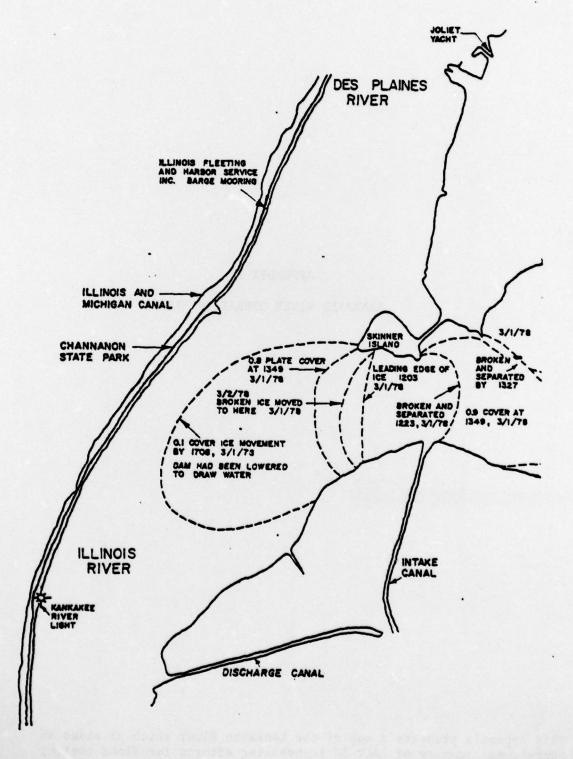


Figure F-la Kankakee River Operation

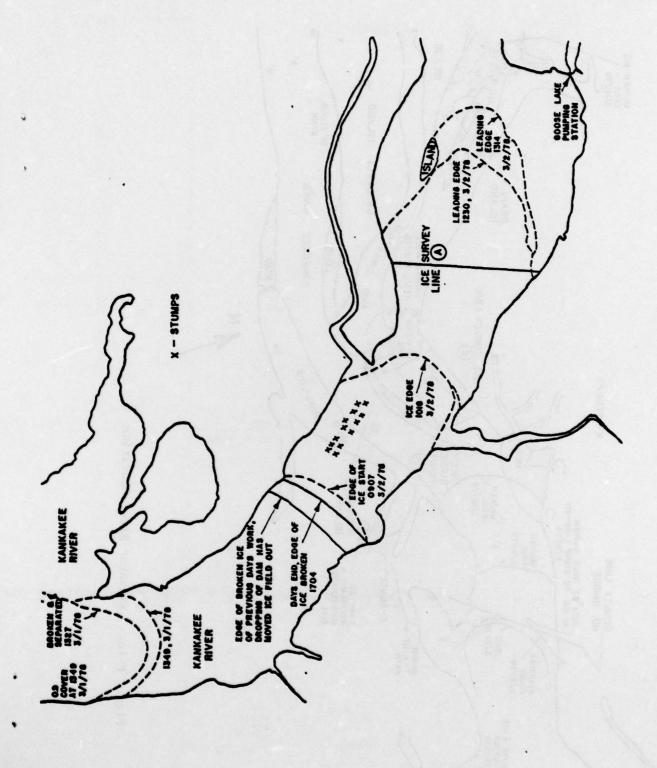


Figure F-1b Kankakee River Operation

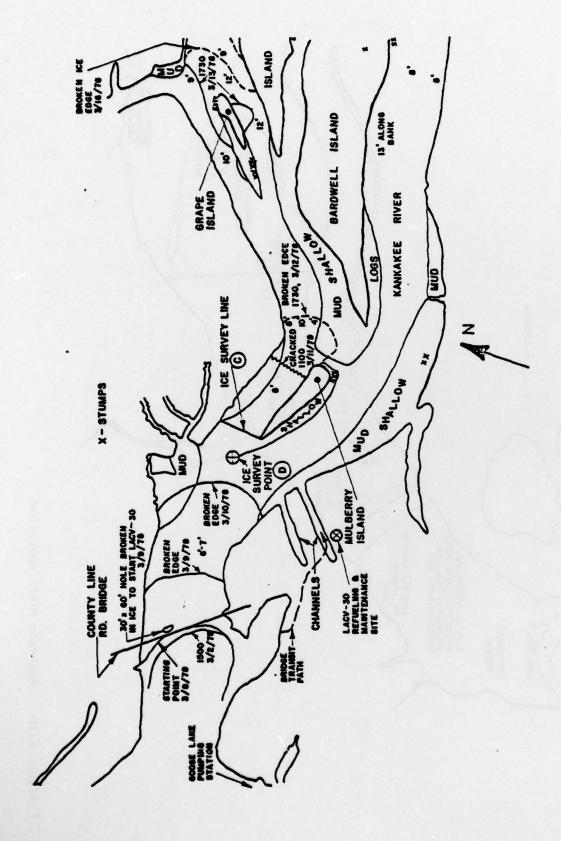


Figure F-1c Kankakee River Operation